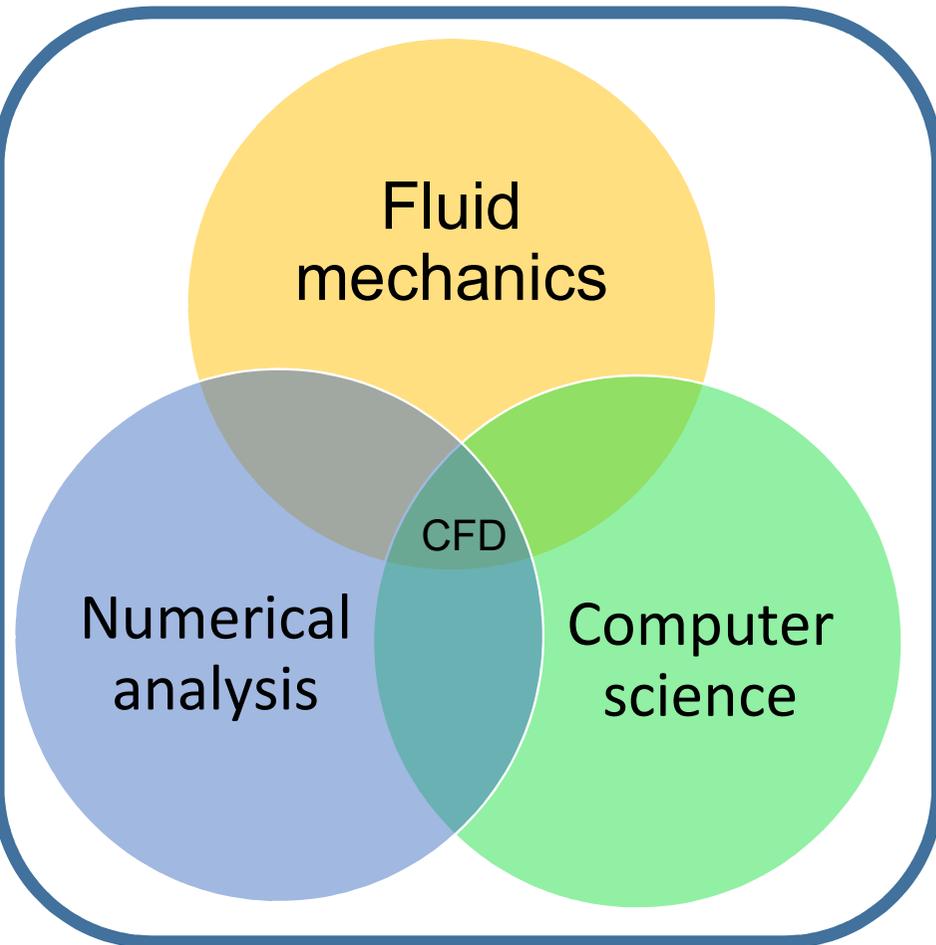


# MEE4006- Computational Fluid Dynamics(CFD)

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# CFD Overview

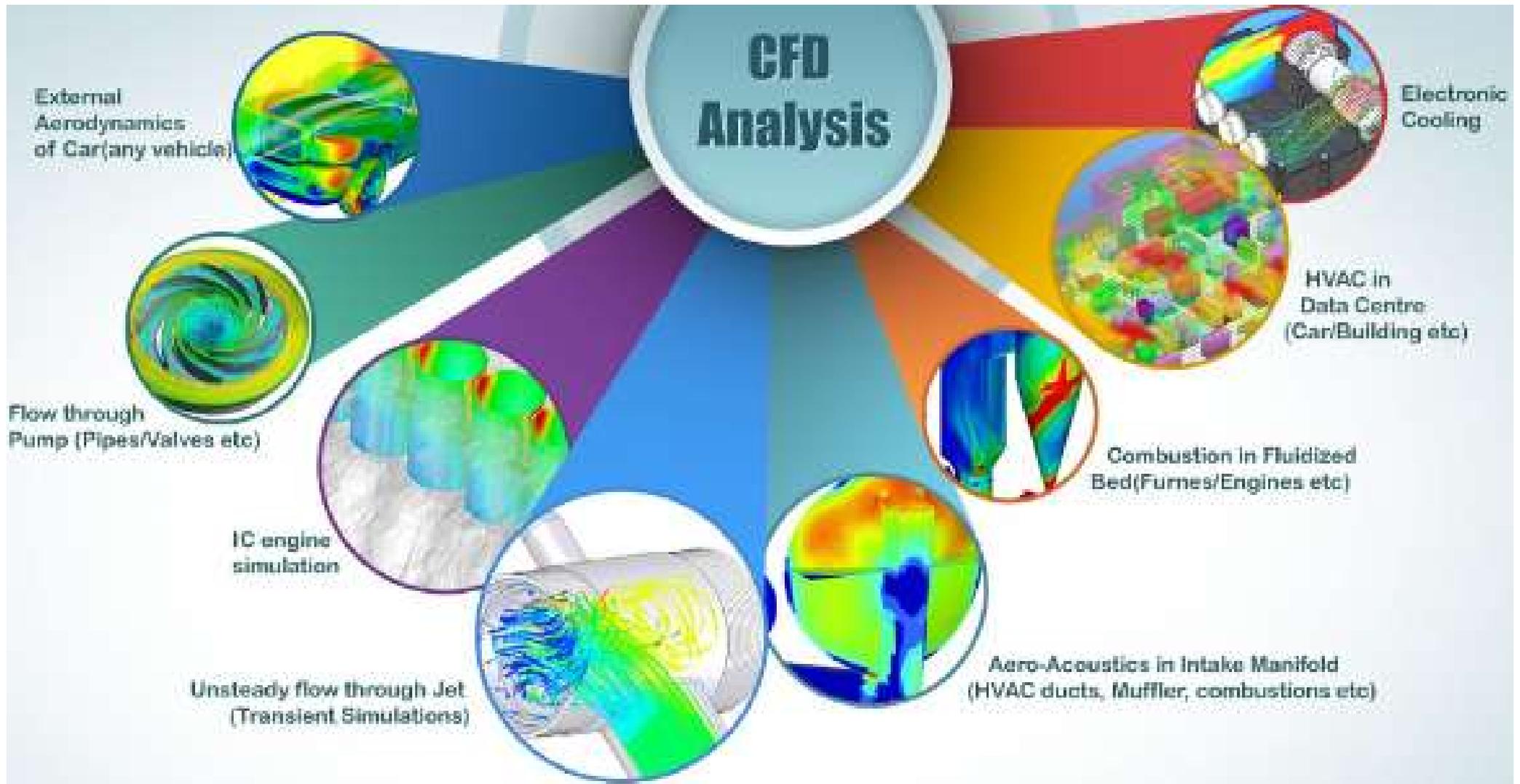


- Lots of university offer courses on CFD and it is an active area of research
- Number of software packages available (e.g. Ansys Fluent)
- Vast literature available on numerical methods for fluid mechanics.
- Widely accepted as a design tool by industrial users
- Even with incompressible flow – impossible to cover everything in single work.
- Based on the speed, the fluid flow is broadly classified into creeping, laminar and turbulent flows.
- Based on the Mach number, fluid flow can be classified into incompressible and compressible flows.
- Type of flow affects the mathematical nature of the problem and therefore the solution method.

The background of the slide is a light blue surface covered with numerous small, glistening water droplets of varying sizes, creating a textured, fresh appearance.

# CFD APPLICATIONS

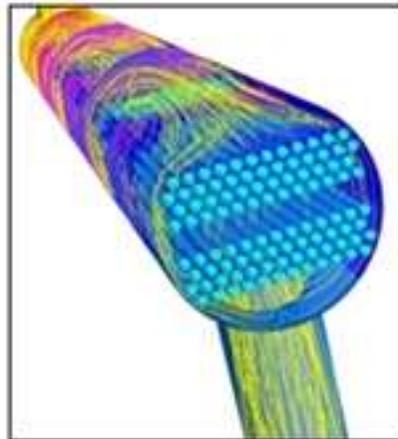
# Wide spectrum of applications



# Materials & Chemical Processing



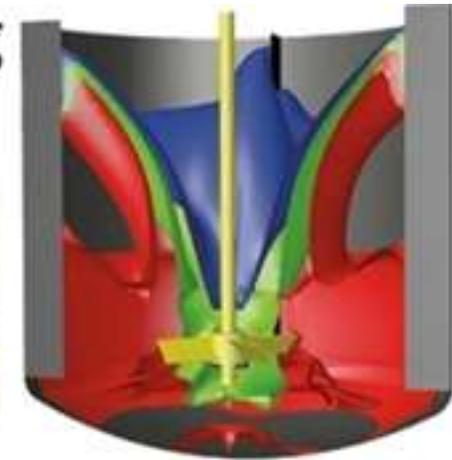
*Chemical sprays*



*Heat exchangers*



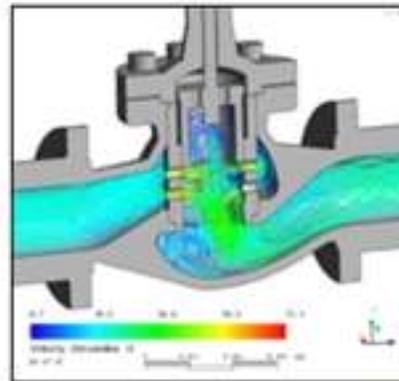
*Dryers*



*Mixing tanks*



*Metal processing*

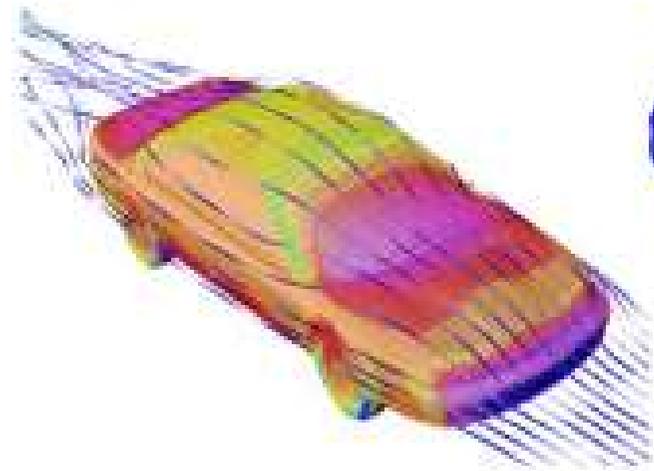


*Valves, flow control*

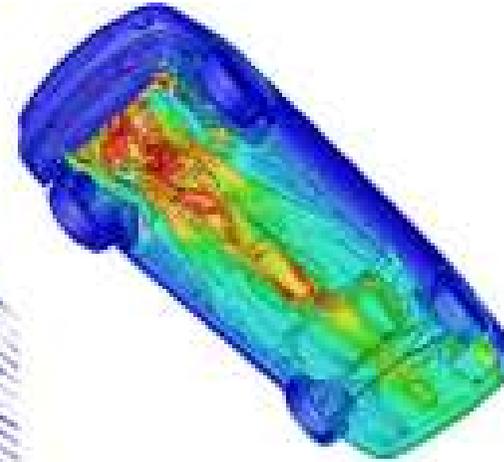


*Separation and filtration*

# Automotive



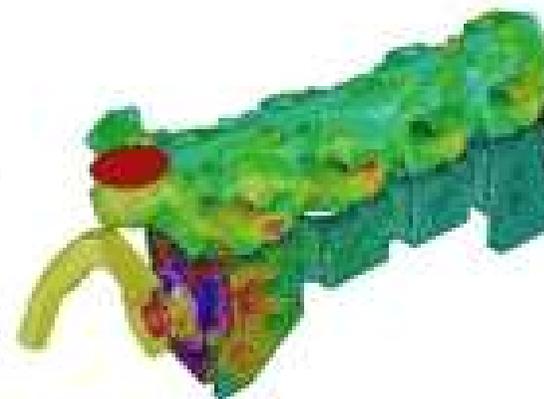
*External Aerodynamics*



*Undercarriage  
Aerodynamics*



*Interior Ventilation*

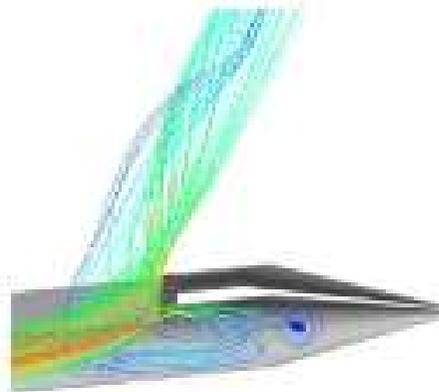


*Engine Cooling*

# Medical



*Medtronic Blood Pump*

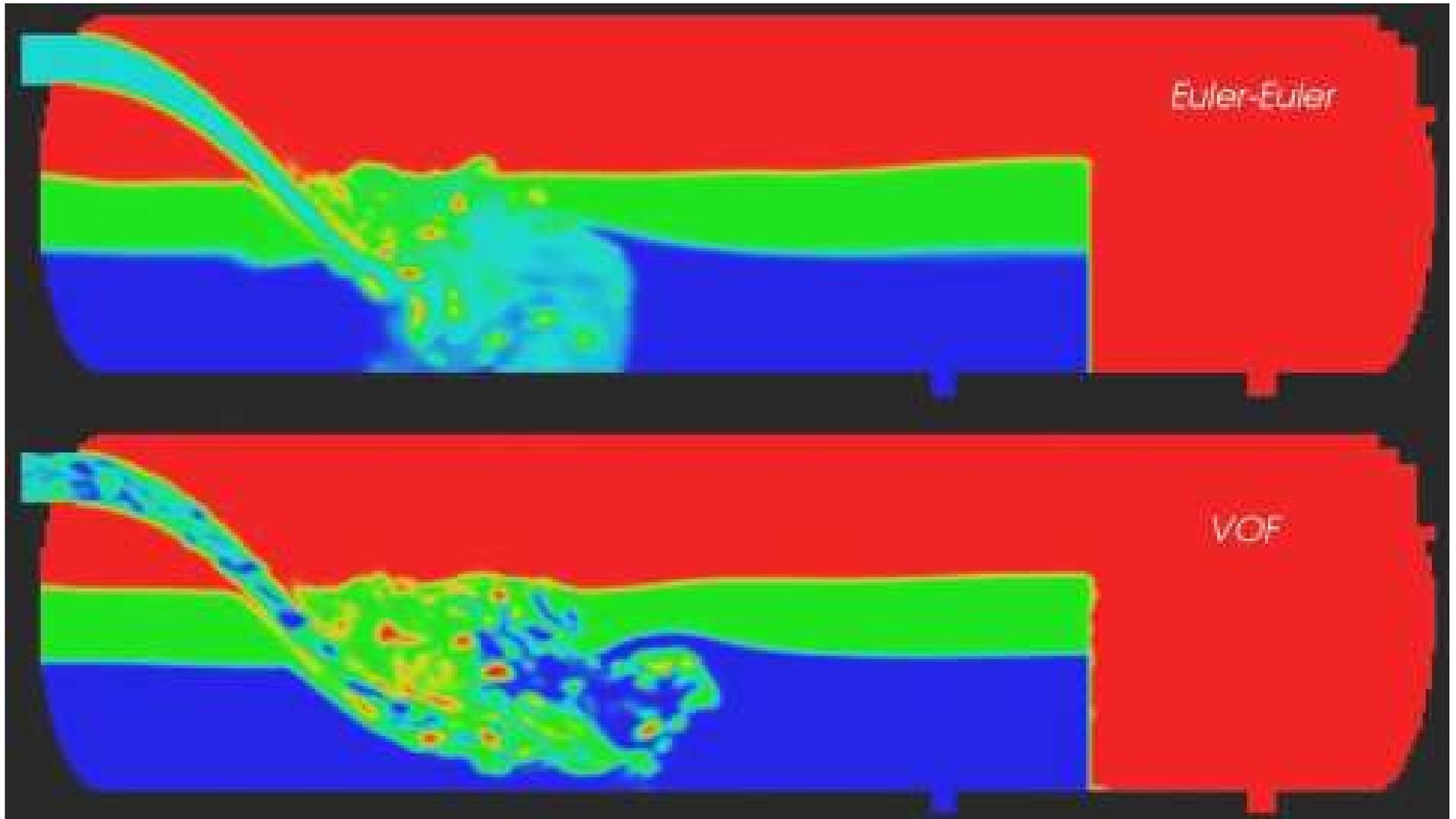


*Spinal Catheter*



*Temperature and natural convection currents in the eye following laser heating.*

# Multiphase flows

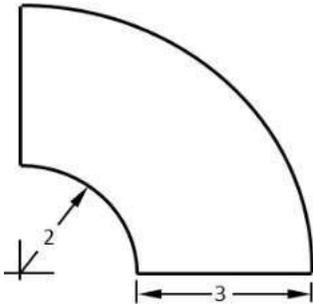


Oil- water separator

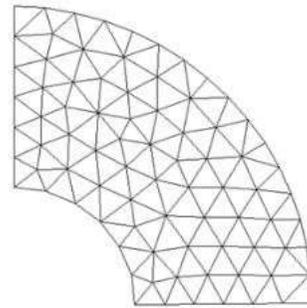


# CFD SIMULATION PROCESS

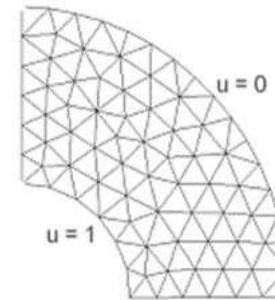
# CFD process- Illustration



1. Build geometry



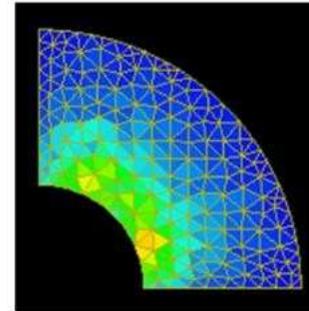
2. Mesh



3. Define boundary conditions

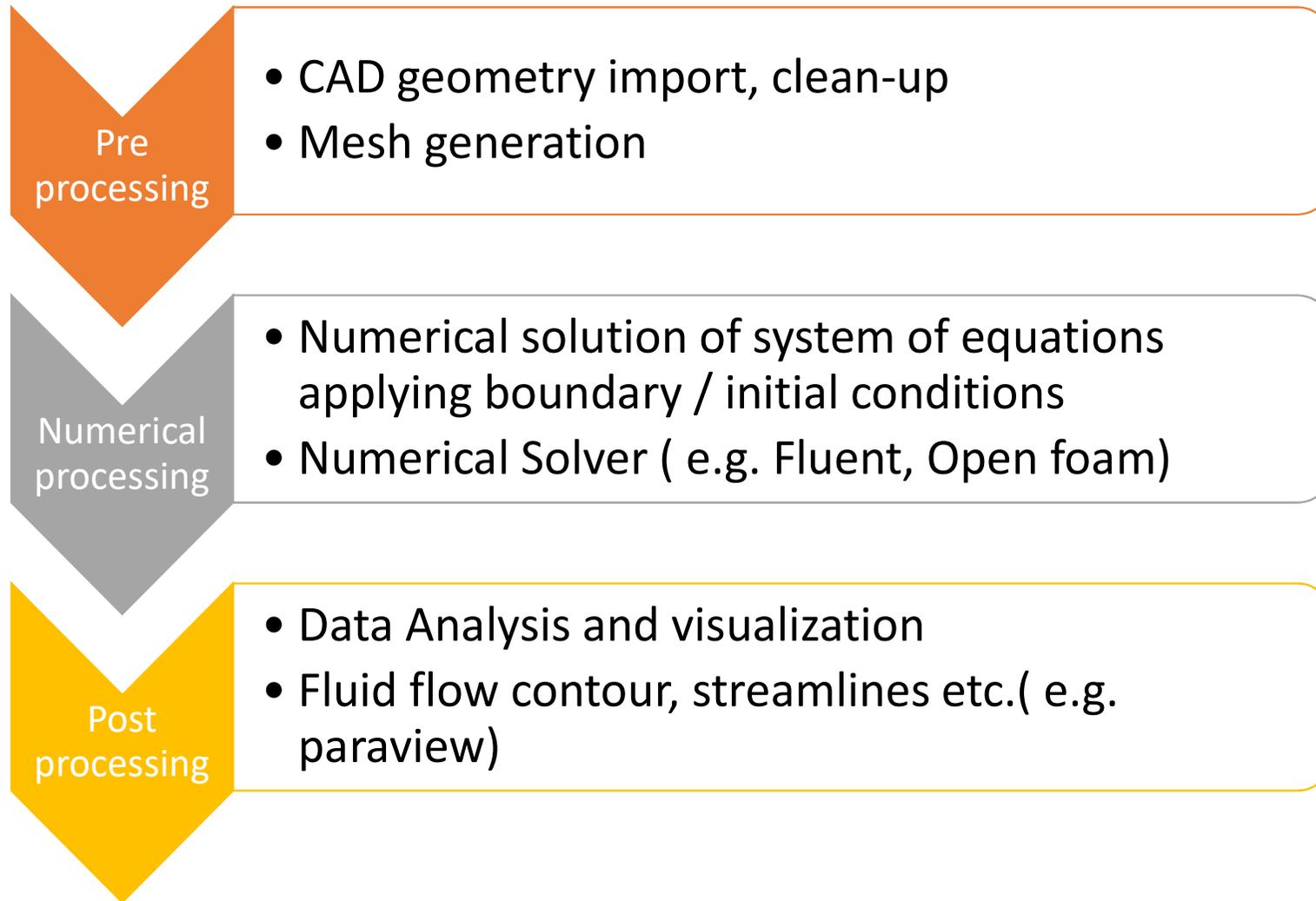


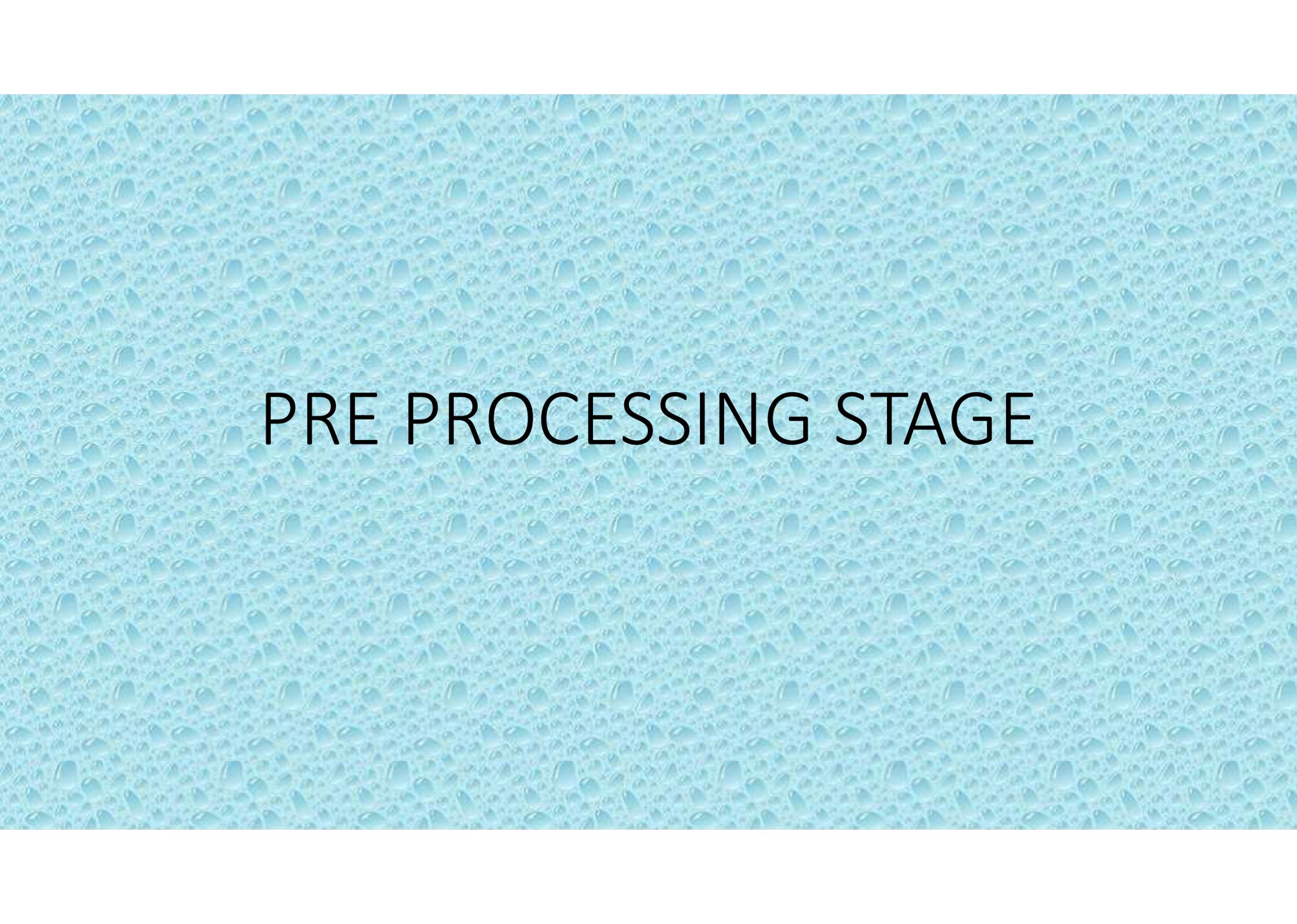
4. Compute



5. Visualize

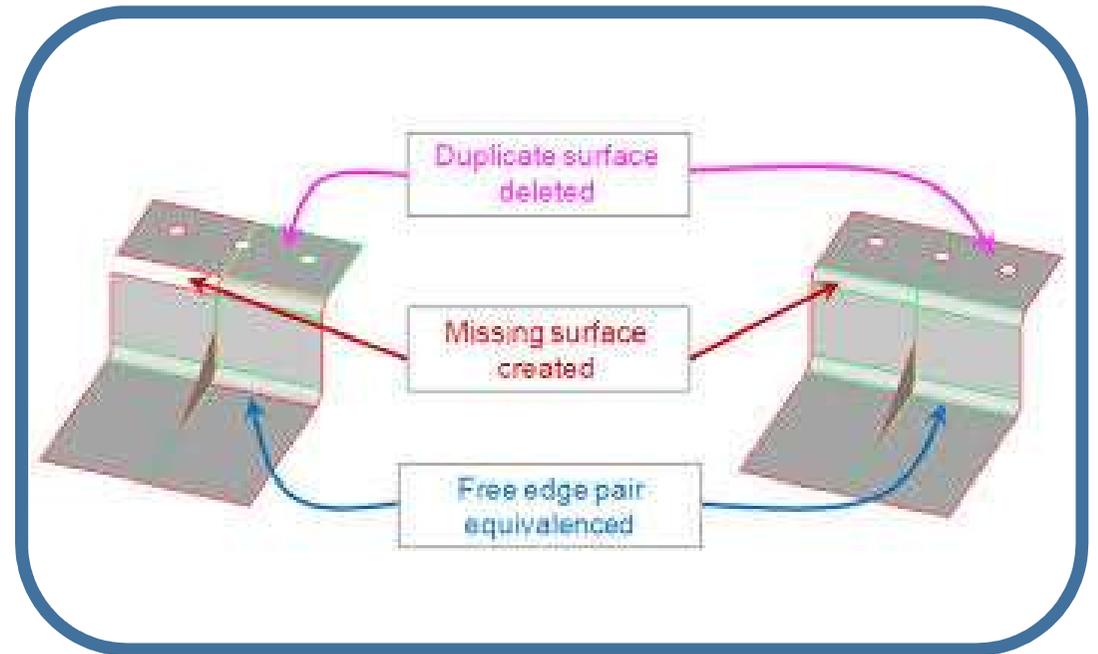
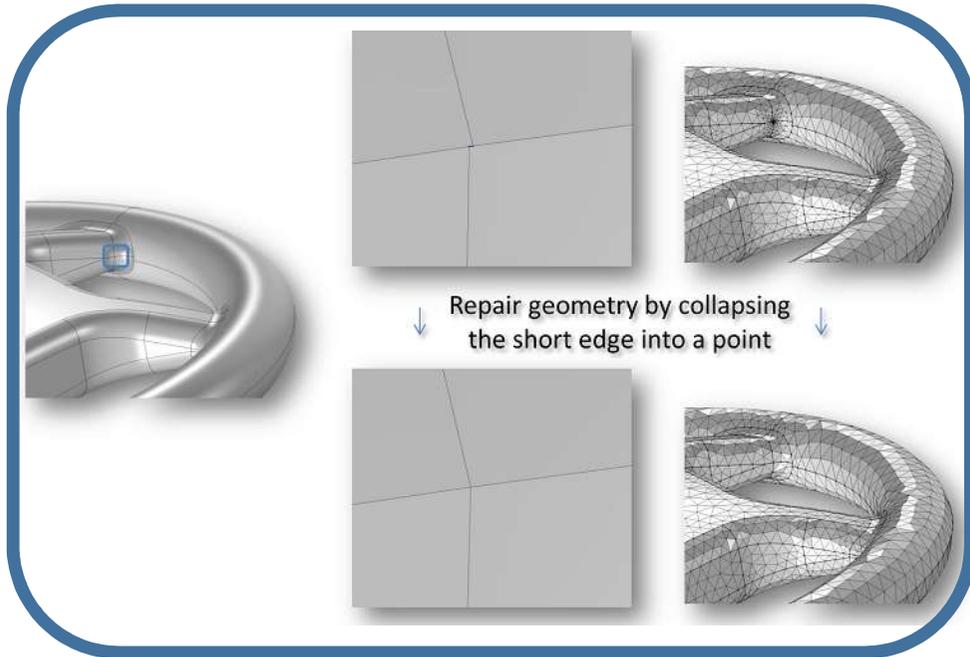
# CFD process- Flow chart



The background of the slide is a light blue surface covered with numerous small, clear water droplets of varying sizes, creating a textured, dew-like appearance. The droplets are more densely packed in some areas and more sparse in others, with some showing highlights and shadows that give them a three-dimensional look.

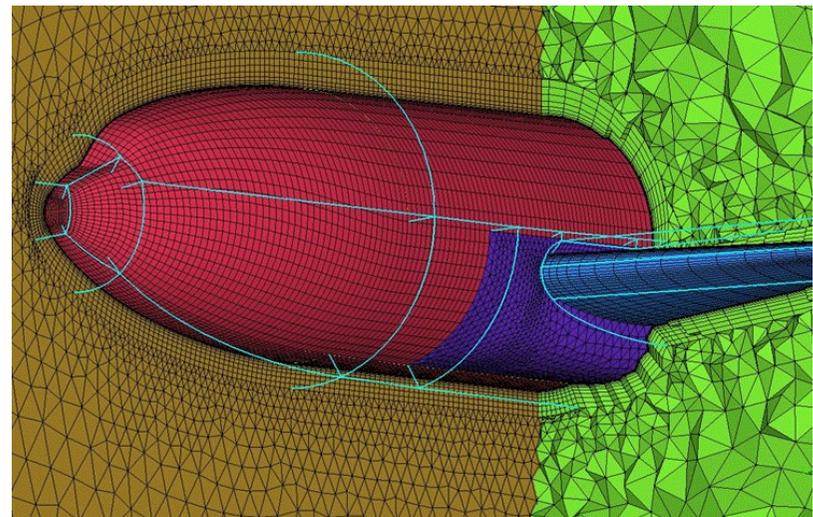
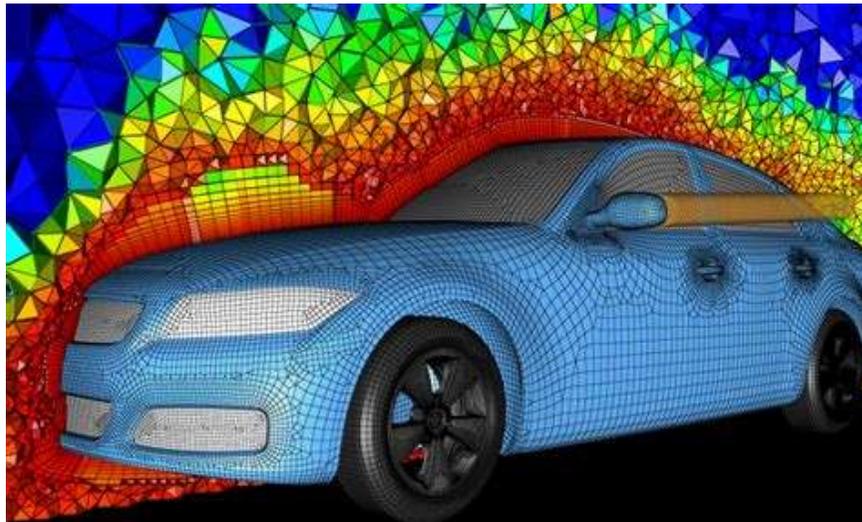
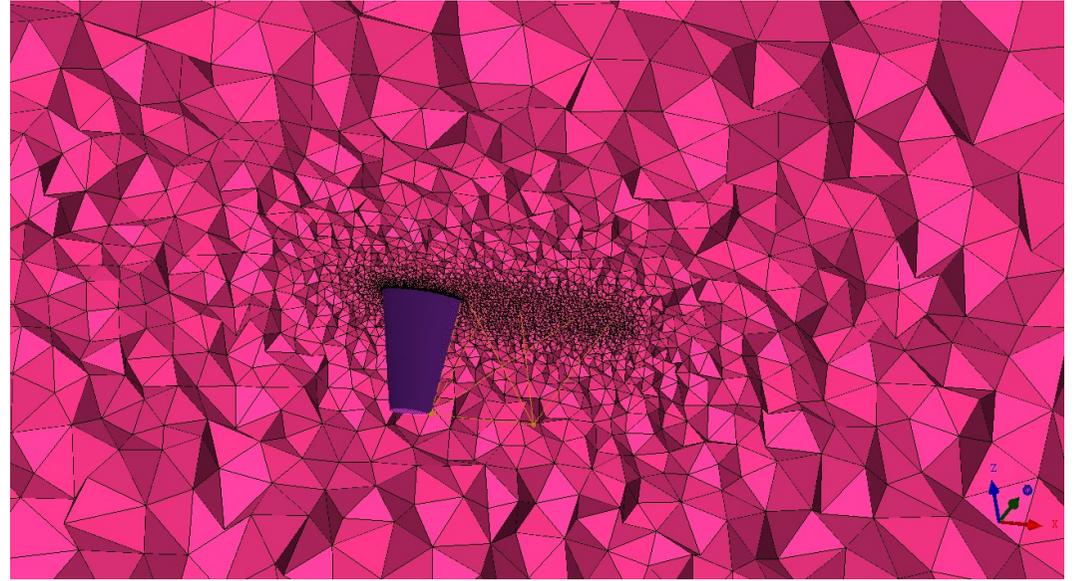
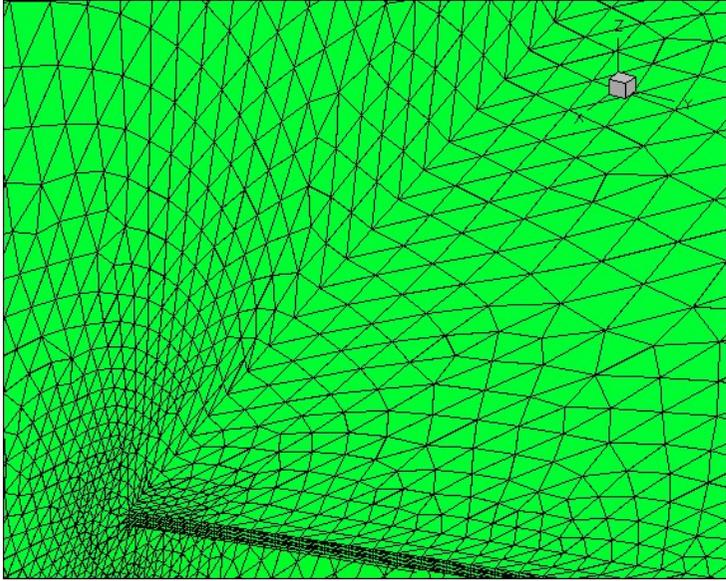
PRE PROCESSING STAGE

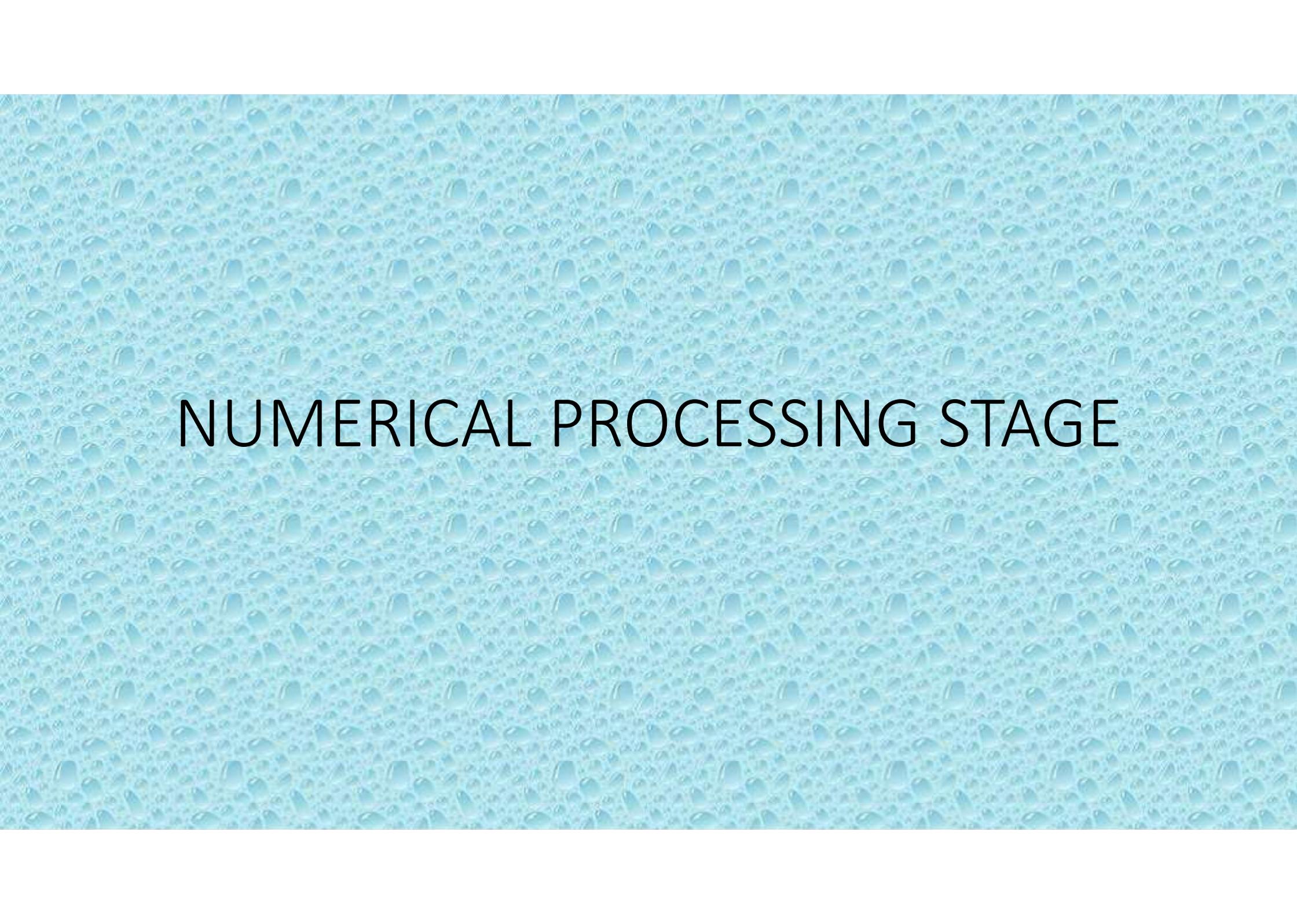
# Geometry import & clean up – An illustration





# Mesh generation – Unstructured mesh





# NUMERICAL PROCESSING STAGE

# Unknowns in the Governing Equations

- In the CFD simulation, it is required to solve numerically a set of Non-linear partial differential equations called the Navier- Stokes Equations.

- For example the governing equations for incompressible flow is given as,

Continuity eq.: 
$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

x-mom.: 
$$\rho\left(\frac{\partial u}{\partial t} + u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} + w\frac{\partial u}{\partial z}\right) = -\frac{\partial p}{\partial x} + \mu\nabla^2 u + \rho g_x$$

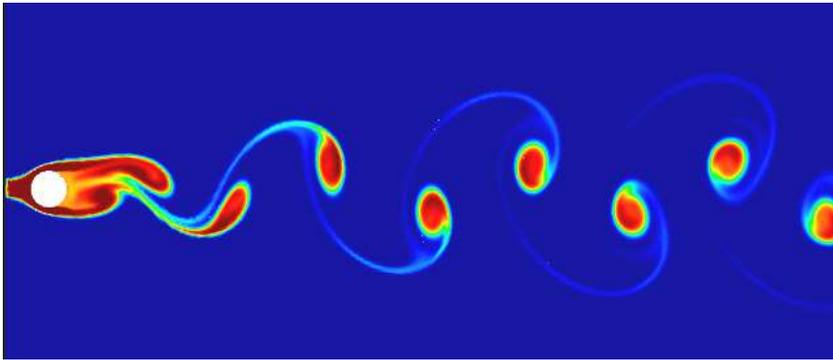
y-mom.: 
$$\rho\left(\frac{\partial v}{\partial t} + u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y} + w\frac{\partial v}{\partial z}\right) = -\frac{\partial p}{\partial y} + \mu\nabla^2 v + \rho g_y$$

z-mom.: 
$$\rho\left(\frac{\partial w}{\partial t} + u\frac{\partial w}{\partial x} + v\frac{\partial w}{\partial y} + w\frac{\partial w}{\partial z}\right) = -\frac{\partial p}{\partial z} + \mu\nabla^2 w + \rho g_z$$

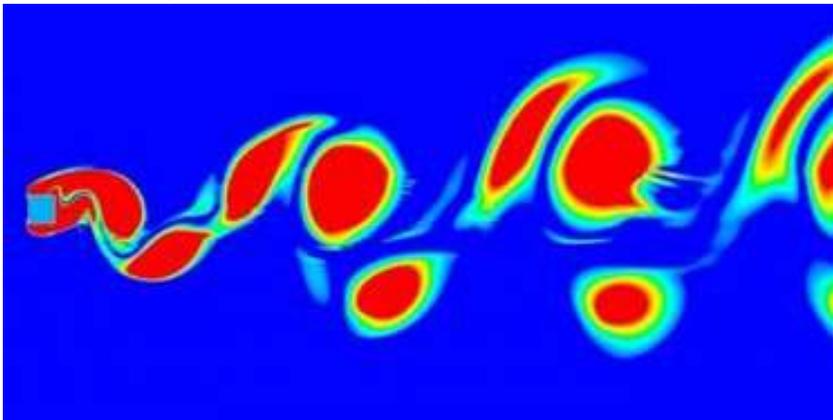
- These equations governs the laws of conservation of mass, momentum.
- The unknown includes the velocity and pressure of the fluid at several discrete points.
- There are several pressure and velocity correction based algorithms available to solve these equations (e.g. SIMPLE)

# Types of fluid flow problems

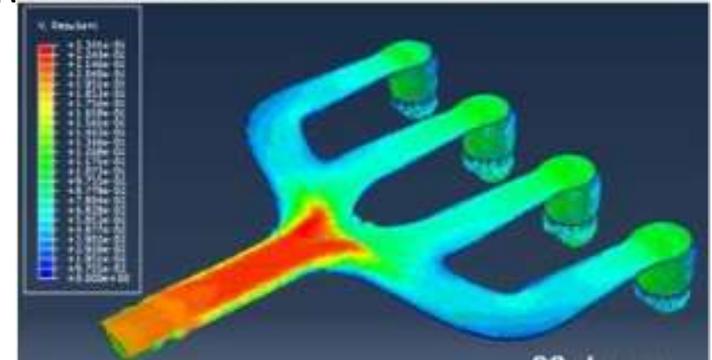
- In the CFD simulation, the fluid flow problems are broadly classified into external and internal flow problems.
- Further classification include steady or unsteady, compressible or incompressible, Laminar or Turbulent flow, one or two or three dimensional flows, natural or forced convection flows



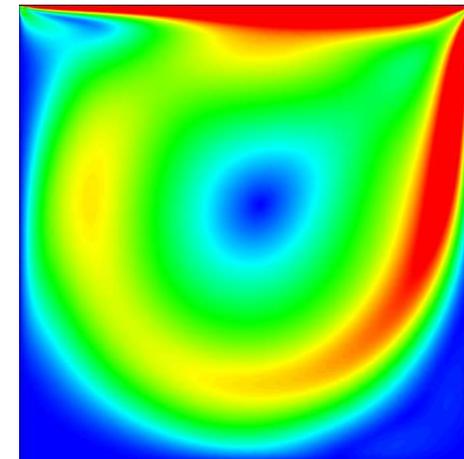
Unsteady external flow past a circular cylinder



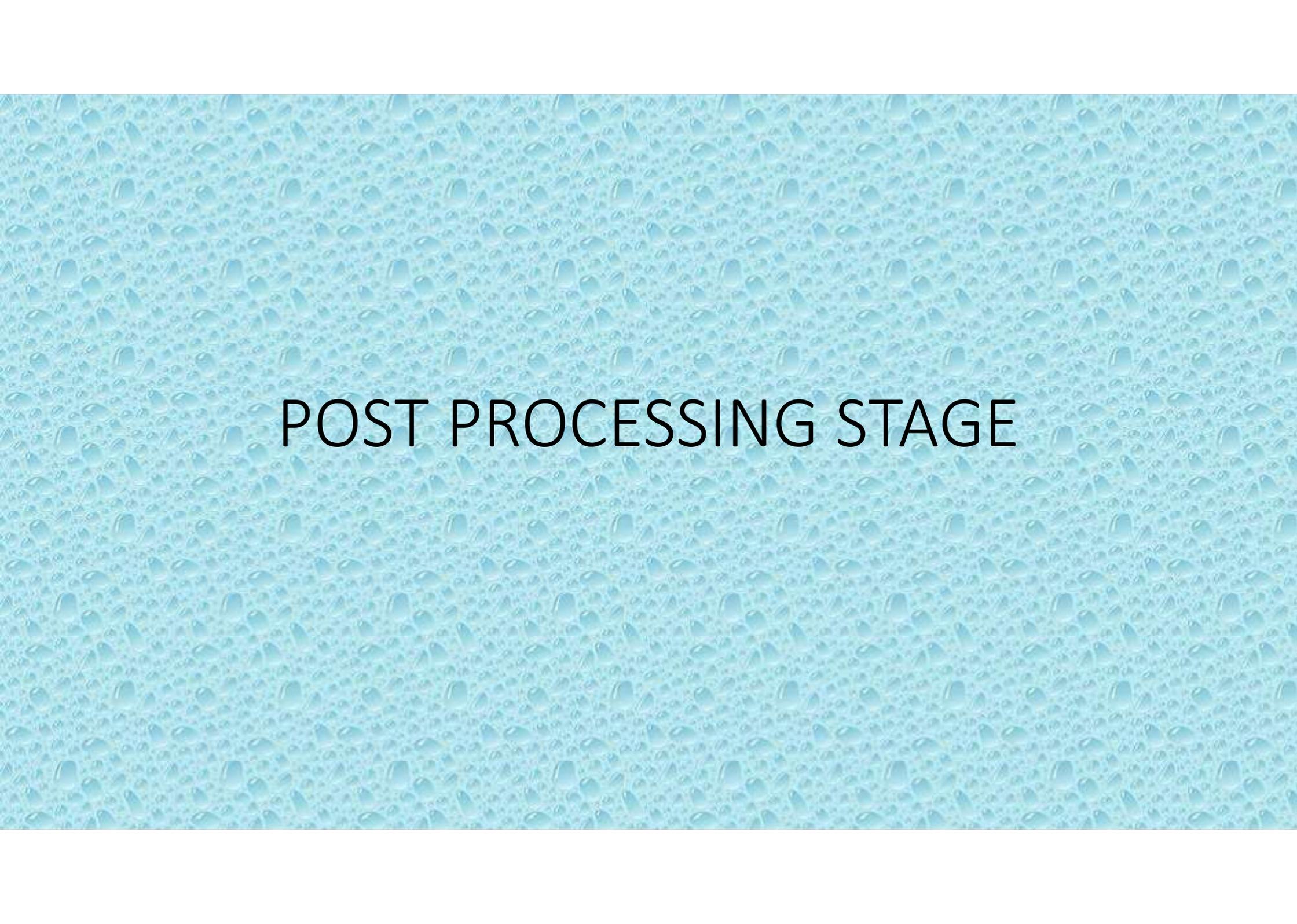
Unsteady external flow past a square cylinder



Internal flow in a pipe

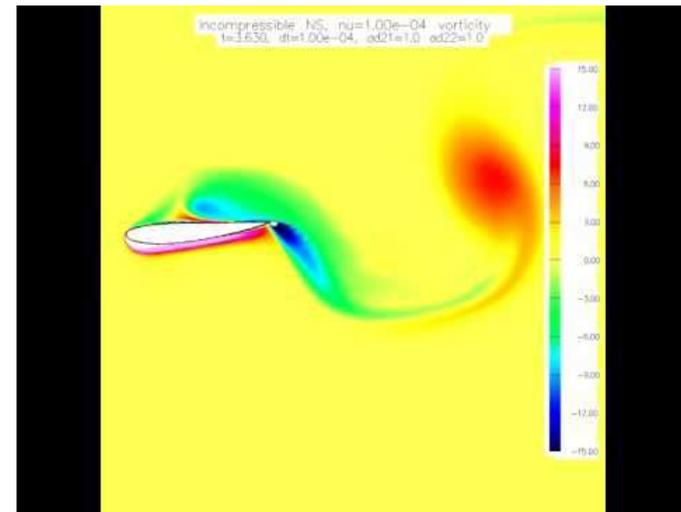
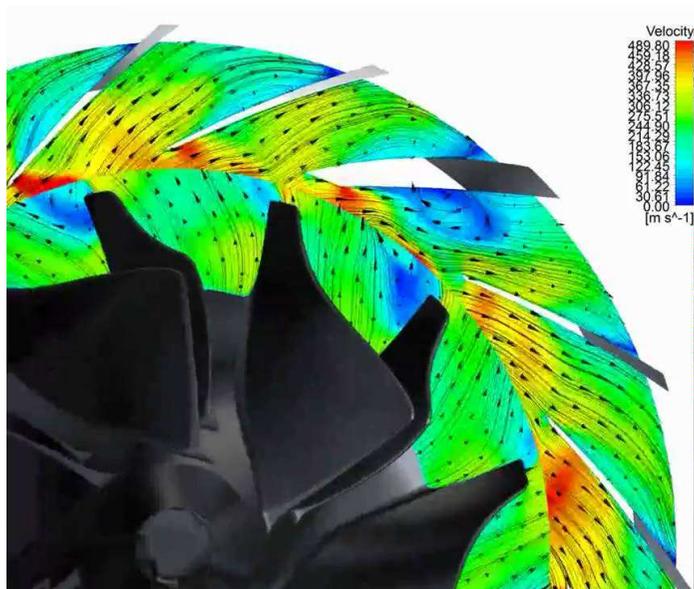
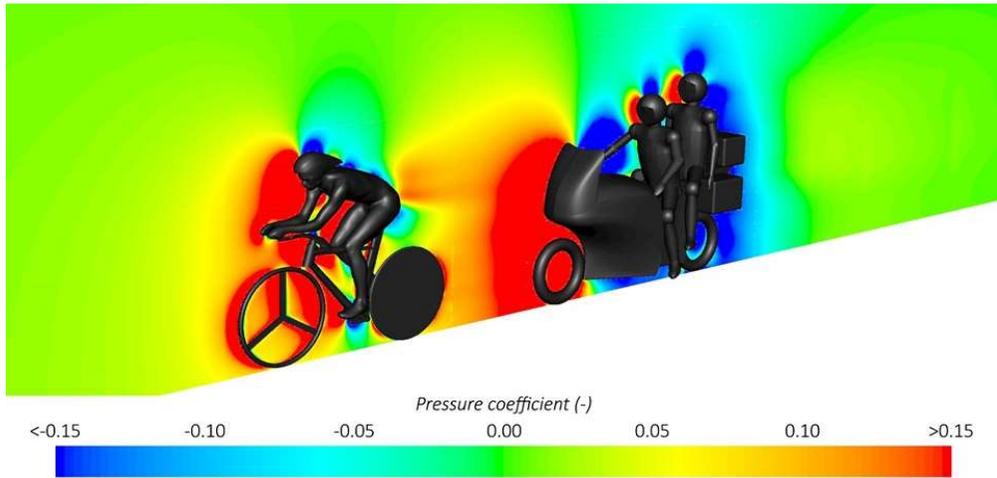


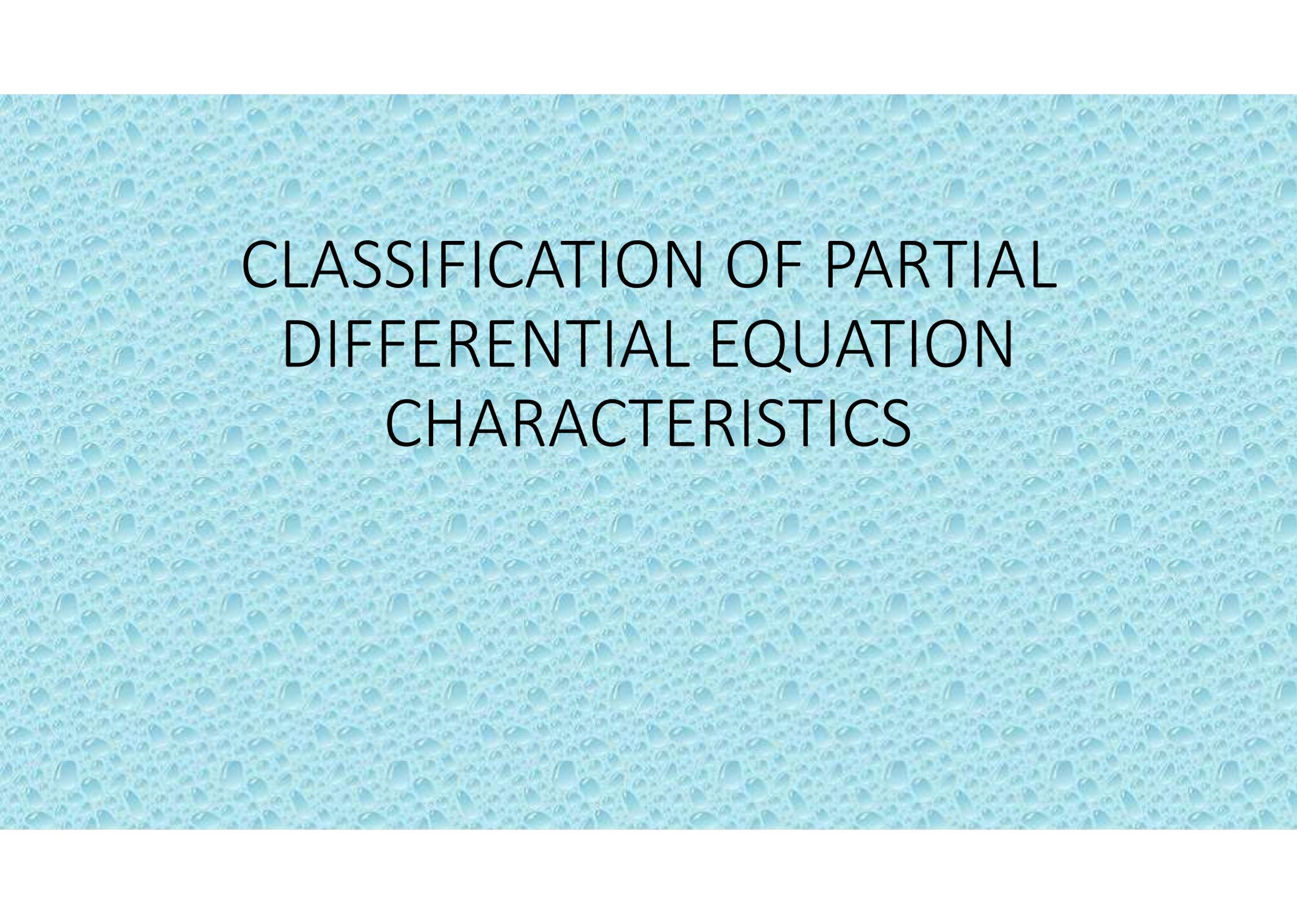
Flow inside a lid driven cavity

The background of the slide is a light blue surface covered with numerous small, clear water droplets of varying sizes, creating a textured, dew-like appearance. The text is centered horizontally and vertically on this background.

POST PROCESSING STAGE

# Data Analysis & Visualization- An Illustration





# CLASSIFICATION OF PARTIAL DIFFERENTIAL EQUATION CHARACTERISTICS

# Characteristics of PDE systems

Consider the linear PDE system

$$A \frac{\partial^2 \phi}{\partial x^2} + B \frac{\partial^2 \phi}{\partial x \partial y} + C \frac{\partial^2 \phi}{\partial y^2} = 0$$

This system is said to be elliptic for the case  $B^2 - 4AC < 0$ .

It is parabolic if  $B^2 - 4AC = 0$ .

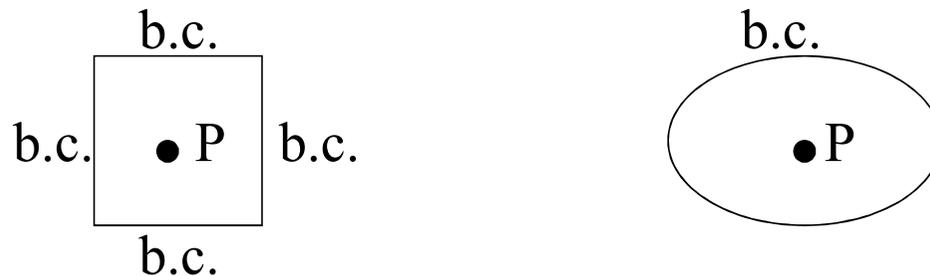
It is hyperbolic when  $B^2 - 4AC > 0$ .

# Elliptic PDE

- Consider steady two dimensional heat conduction governed by the equation

$$k \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) + Q = 0$$

- Here,  $A = C = k$  and  $B = 0$ . Hence  $B^2 - 4AC = -4k^2 < 0$ .
- Therefore, the system is elliptic.
- For an elliptic PDE, the boundary conditions need to be given on a closed boundary.
- In other words, the boundary conditions all around influence the solution at a point



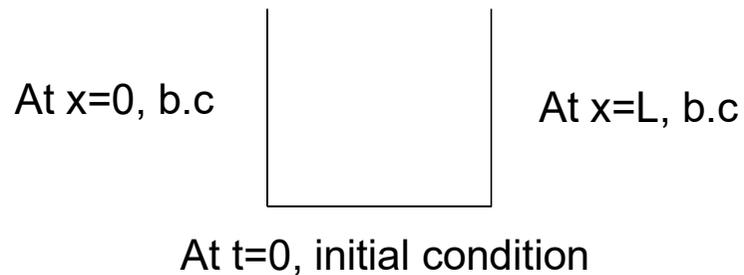
Boundary conditions for elliptic systems

# Parabolic PDE

- Transient heat conduction problem which follows the governing equation

$$\frac{\partial T}{\partial t} = \frac{k}{\rho c} \frac{\partial^2 T}{\partial x^2} = \alpha \frac{\partial^2 T}{\partial x^2}$$

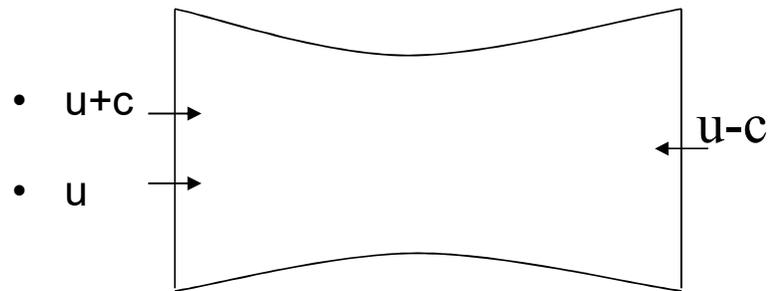
- Here,  $A = \alpha$ ,  $B = 0$  and  $C = 0$ .
- Hence,  $B^2 - 4AC = 0$
- It is a parabolic system.
- For a parabolic system the conditions need to be specified as shown below.



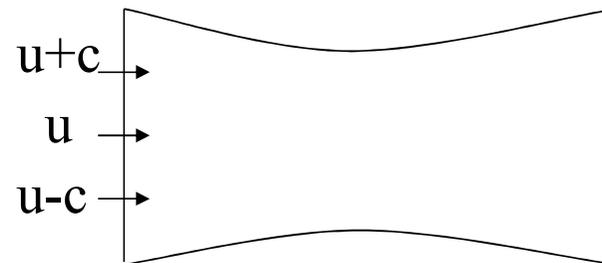
# Hyperbolic PDE

- The wave equation  $\frac{\partial^2 y}{\partial t^2} = c^2 \frac{\partial^2 y}{\partial x^2}$  is a hyperbolic system, with  $c$  denoting the acoustic speed.
- Here,  $B = 0$  and  $A = 1$ ,  $C = -c^2$ .
- Hence,  $B^2 - 4AC = 0 - 4 \times 1 \times (-c^2) = 4c^2 > 0$ .
- For a hyperbolic system, there are characteristic variables which determine the number of boundary conditions to be given.
- In the above case, the two characteristics  $(x + ct)$  and  $(x - ct)$  represent the solutions corresponding to the backward-and forward- propagating waves.

# Boundary conditions for hyperbolic PDE



Subsonic flow



Supersonic flow

- A compressible flow has three characteristic velocities i.e.  $u+c$ ,  $u$ ,  $u-c$ .
- Depending on the number of characteristics crossing into the domain at the boundary, the b.c. are decided.

BASIC ASPECTS OF NUMERICAL  
DISCRETIZATION METHODS FOR  
PARTIAL DIFFERENTIAL  
EQUATION (PDE)

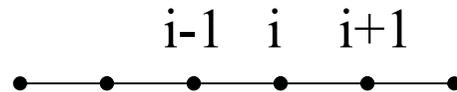
# Types of Numerical Discretization Techniques

- FINITE DIFFERENCE METHOD
- FINITE VOLUME METHOD
- FINITE ELEMENT METHOD
- BOUNDARY ELEMENT METHOD
- SPECTRAL METHOD

# Finite Difference Method

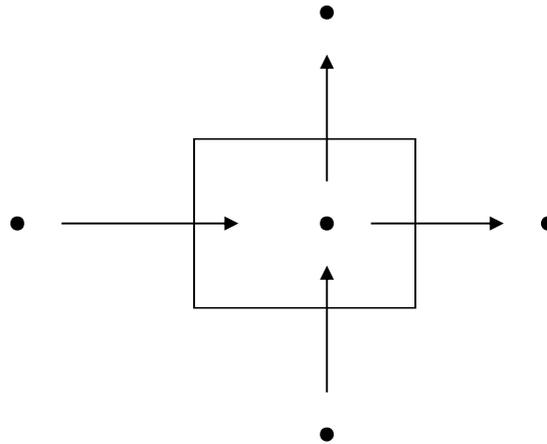
- In this method, differential equations are converted into difference expressions

$$\frac{dT}{dx} = \frac{T_i - T_{i-1}}{\Delta x} \quad \text{or} \quad \frac{T_{i+1} - T_i}{\Delta x}$$



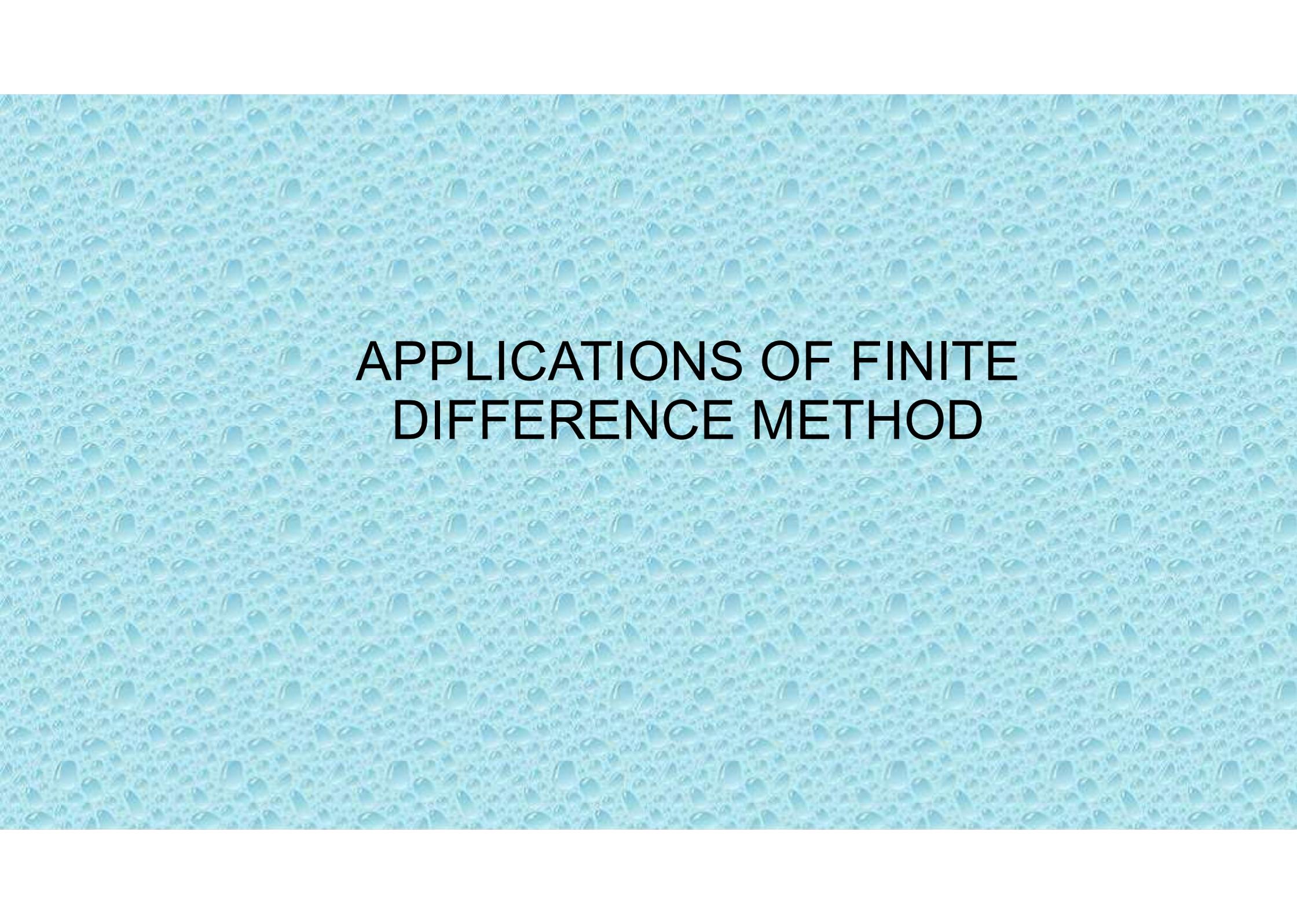
# Finite Volume Method

- Flux balance is applied for each cell.
- Heat flux in – Heat flux out = rate of thermal storage
- Fluxes are approximated using neighboring nodes



# Finite Element Method

- While FDM & FVM are applied for flow/thermal problems, FEM was initially developed for structural problems.
- In this method, a large structure is divided into small elements and characteristic of each element is written as a matrix contribution.
- By adding contributions of all elements, we get matrix equation for the whole geometry.



# APPLICATIONS OF FINITE DIFFERENCE METHOD

# Taylor Series Expansions

$$T_{i-1} = T_i - \left( \frac{dT}{dx} \right)_i \Delta x + \left( \frac{d^2 T}{dx^2} \right)_i \frac{\Delta x^2}{2!} - \left( \frac{d^3 T}{dx^3} \right)_i \frac{\Delta x^3}{3!} + \dots + \left( \frac{d^n T}{dx^n} \right)_i \frac{(-\Delta x)^n}{n!} + O(\Delta x^{n+1})$$

$$T_{i+1} = T_i + \left( \frac{dT}{dx} \right)_i \Delta x + \left( \frac{d^2 T}{dx^2} \right)_i \frac{\Delta x^2}{2!} + \left( \frac{d^3 T}{dx^3} \right)_i \frac{\Delta x^3}{3!} + \dots + \left( \frac{d^n T}{dx^n} \right)_i \frac{\Delta x^n}{n!} + O(\Delta x^{n+1})$$

$$T_{i+2} = T_i + \left( \frac{dT}{dx} \right)_i (2\Delta x) + \left( \frac{d^2 T}{dx^2} \right)_i \frac{(2\Delta x)^2}{2!} + \left( \frac{d^3 T}{dx^3} \right)_i \frac{(2\Delta x)^3}{3!} \\ \dots + \left( \frac{d^n T}{dx^n} \right)_i \frac{(2\Delta x)^n}{n!} + O(\Delta x^{n+1})$$

$$T_{i-2} = T_i - \left( \frac{dT}{dx} \right)_i (2\Delta x) + \left( \frac{d^2 T}{dx^2} \right)_i \frac{(2\Delta x)^2}{2!} - \left( \frac{d^3 T}{dx^3} \right)_i \frac{(2\Delta x)^3}{3!} \\ + \left( \frac{d^n T}{dx^n} \right)_i \frac{(-2\Delta x)^n}{n!} + O(\Delta x^{n+1})$$

# Derivative Approximations

$$\begin{aligned}\left(\frac{dT}{dx}\right)_i &= \frac{T_{i+1} - T_i}{\Delta x} - \left(\frac{d^2 T}{dx^2}\right)_i \frac{\Delta x}{2!} - \left(\frac{d^3 T}{dx^3}\right)_i \frac{\Delta x^2}{3!} \\ &= \frac{T_{i+1} - T_i}{\Delta x} + O(\Delta x)\end{aligned}$$

$$\begin{aligned}\left(\frac{dT}{dx}\right)_i &= \frac{T_i - T_{i-1}}{\Delta x} - \left(\frac{d^2 T}{dx^2}\right)_i \frac{\Delta x}{2!} + \left(\frac{d^3 T}{dx^3}\right)_i \frac{\Delta x^2}{3!} \\ &= \frac{T_i - T_{i-1}}{\Delta x} + O(\Delta x)\end{aligned}$$

$$\left(\frac{dT}{dx}\right)_i = \frac{T_{i+1} - T_{i-1}}{2 \Delta x} + O(\Delta x^2)$$

# Derivative Approximation

$$4T_{i+1} - T_{i+2} = 3T_i + 2\left(\frac{dT}{dx}\right)_i \Delta x + 0(\Delta x^3)$$

$$\left(\frac{dT}{dx}\right)_i = \frac{4T_{i+1} - T_{i+2} - 3T_i}{2\Delta x} + 0(\Delta x^2)$$

$$T_{i+1} + T_{i-1} = 2T_i + 2\left(\frac{d^2T}{dx^2}\right)_i \frac{\Delta x^2}{2!} + 2\left(\frac{d^4T}{dx^4}\right)_i \frac{\Delta x^4}{4!} + \dots$$

$$\left(\frac{d^2T}{dx^2}\right)_i = \frac{T_{i+1} + T_{i-1} - 2T_i}{\Delta x^2} + 0(\Delta x^2)$$

# Estimation of Error

$$\varepsilon_i^k = T(x_i, t^k) - T^*(x_i, t^k)$$

$$\varepsilon_i^k \propto \Delta x_i^2 \quad \text{and} \quad \varepsilon_i^k \propto \Delta t^k$$

$$\varepsilon = O(\Delta x^2, \Delta t)$$

**NUMERICAL ALGORITHM TO SOLVE  
NAVIER STOKES EQUATION-  
PRESSURE CORRECTION  
APPROACH**

# VELOCITY-PRESSURE FORMULATION

CONTINUITY EQUATION

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

X-MOMENTUM EQ. (FOR UPDATING U VELOCITY):

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = - \frac{1}{\rho} \frac{\partial p}{\partial x} + \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\}$$

Y-MOMENTUM EQ. (FOR UPDATING V VELOCITY):

$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = - \frac{1}{\rho} \frac{\partial p}{\partial y} + \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\}$$

# SIMPLE METHOD

## Semi- IMplicit Pressure Linked Equation Solver-- SIMPLE

X-mom.: 
$$\frac{\partial u}{\partial t} = - \left( u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \Big| ^n - \frac{1}{\rho} \frac{\partial p}{\partial x} \Big| ^{n+1} + \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\} \Big| ^n$$

$$u^{n+1} = u^n - \Delta t. \left( u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \Big| ^n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial x} \Big| ^{n+1} + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\} \Big| ^n$$

Y-mom.: 
$$\frac{\partial v}{\partial t} = - \left( u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \Big| ^n - \frac{1}{\rho} \frac{\partial p}{\partial y} \Big| ^{n+1} + \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\} \Big| ^n$$

$$v^{n+1} = v^n - \Delta t. \left( u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \Big| ^n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial y} \Big| ^{n+1} + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\} \Big| ^n$$

# VELOCITY CORRECTION EQUATION- x momentum

$$u^{n+1} = u^n - \Delta t. \left( u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \Big|_n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial x} \Big|^{n+1} + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\} \Big|_n$$

$$u^* = u^n - \Delta t. \left( u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \Big|_n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial x} \Big|_n^* + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\} \Big|_n$$

$$u^{n+1} - u^* = -\Delta t. \left( \frac{1}{\rho} \frac{\partial p}{\partial x} \right) \Big|_n^{n+1} + \Delta t. \left( \frac{1}{\rho} \frac{\partial p}{\partial x} \right) \Big|_n^*$$

# VELOCITY CORRECTION EQUATION- y momentum

$$v^{n+1} = v^n - \Delta t. \left( u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \Big|_n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial y} \Big|^{n+1} + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\} \Big|_n$$

$$v^* = v^n - \Delta t. \left( u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \Big|_n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial y} \Big|_* + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\} \Big|_n$$

$$v^{n+1} - v^* = -\Delta t. \left( \frac{1}{\rho} \frac{\partial p}{\partial y} \right)^{n+1} + \Delta t. \left( \frac{1}{\rho} \frac{\partial p}{\partial y} \right)^*$$

# PRESSURE CORRECTIONS

Define

$$u' = u^{n+1} - u^* \quad v' = v^{n+1} - v^* \quad p' = p^{n+1} - p^*$$

It can be shown that

$$u' = -\frac{\Delta t}{\rho} \frac{\partial p'}{\partial x} \quad v' = -\frac{\Delta t}{\rho} \frac{\partial p'}{\partial y}$$

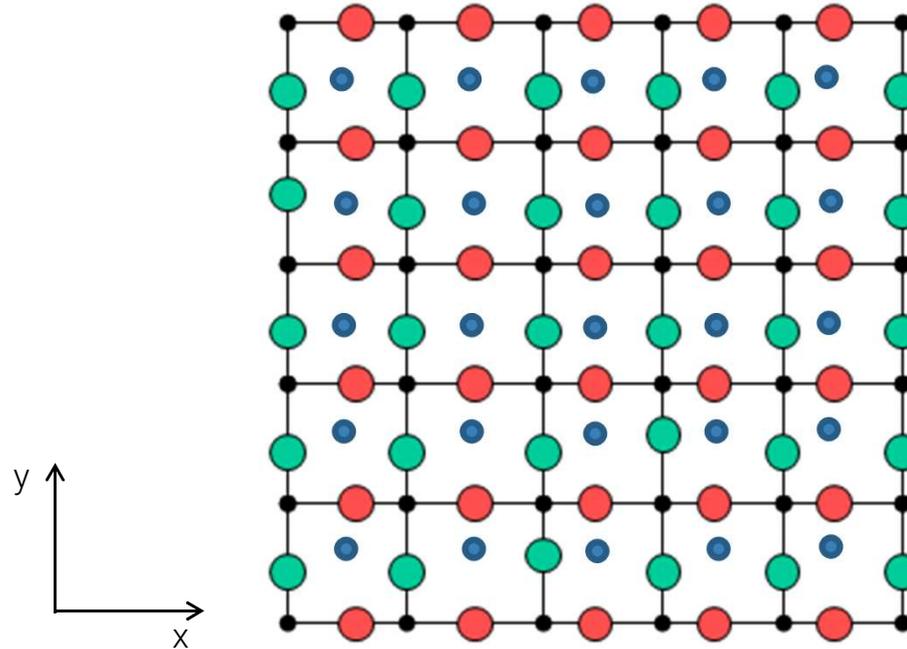
Substituting for velocity & pressure corrections, we get

$$\frac{\partial^2 p'}{\partial x^2} + \frac{\partial^2 p'}{\partial y^2} = -\frac{\rho}{\Delta t} \left( \frac{\partial u'}{\partial x} + \frac{\partial v'}{\partial y} \right) = \frac{\rho}{\Delta t} \left( \frac{\partial u^*}{\partial x} + \frac{\partial v^*}{\partial y} \right)$$

# Step Involved In SIMPLE

- At the start of a time step, assume a guess pressure field  $p^*$
- Solve momentum equations to get guess velocities  $u^*$  and  $v^*$  at each node
- Using  $u^*$  and  $v^*$  calculate continuity residue at each point
- From continuity equation residue, solve for pressure correction  $p'$  at each node
- Using  $p'$  solve for velocity corrections
- Update variables as  $p^{n+1}=p^*+p'$ ,  $u^{n+1}=u^*+u'$ ,  $v^{n+1}=v^*+v'$
- And go to next time step

# Staggered & Collocated Mesh



	Staggered	Semi-Staggered	Collocated
●	V- velocity	V- velocity	-
●	U- velocity	U- velocity	-
●	Cell vertices	Pressure (Cell vertices)	-
●	Pressure (Cell centers)	Cell centers	U,V-velocities, Pressure

# Staggered Mesh Procedure

- Pressure nodes are taken as the main nodes.
- x-velocity ( $u$ ) nodes are shifted by  $dx/2$  with reference to pressure nodes .
- and y-velocity ( $v$ ) nodes are shifted by  $dy/2$  with reference to pressure nodes.
- Such a staggered mesh avoids odd-even decoupling (chequer-board configuration) between velocities & pressures .