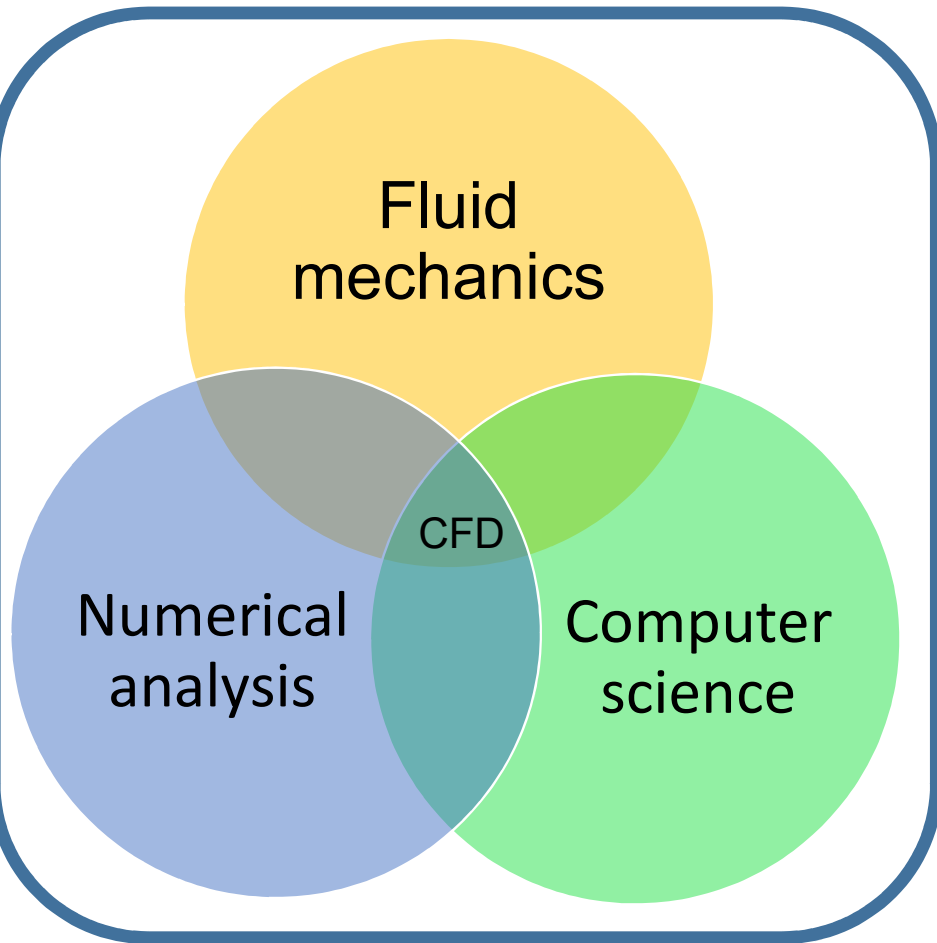


MEE4006- Computational Fluid Dynamics(CFD)

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Associate Professor,
School of Mechanical Engineering (SMEC),
VIT University,
Vellore-632014, Tamilnadu, India

CFD Overview

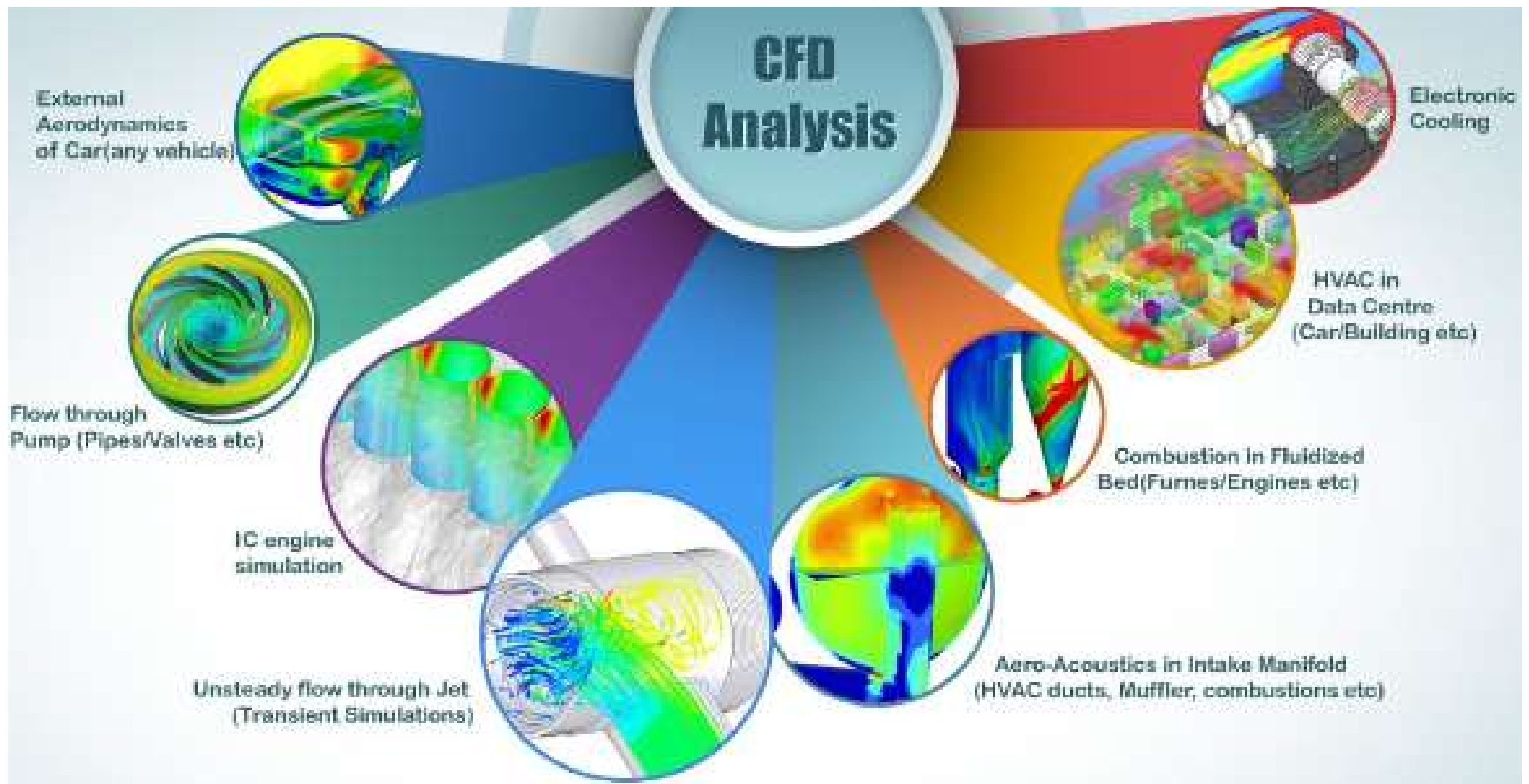


- Lots of university offer courses on CFD and it is an active area of research
- Number of software packages available (e.g. Ansys Fluent)
- Vast literature available on numerical methods for fluid mechanics.
- Widely accepted as a design tool by industrial users
- Even with incompressible flow – impossible to cover everything in single work.
- Based on the speed, the fluid flow is broadly classified into creeping, laminar and turbulent flows.
- Based on the Mach number, fluid flow can be classified into incompressible and compressible flows.
- Type of flow affects the mathematical nature of the problem and therefore the solution method.



CFD APPLICATIONS

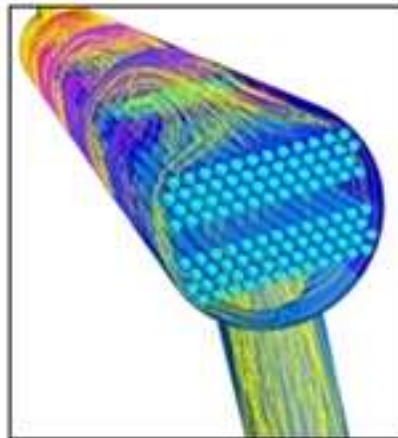
Wide spectrum of applications



Materials & Chemical Processing



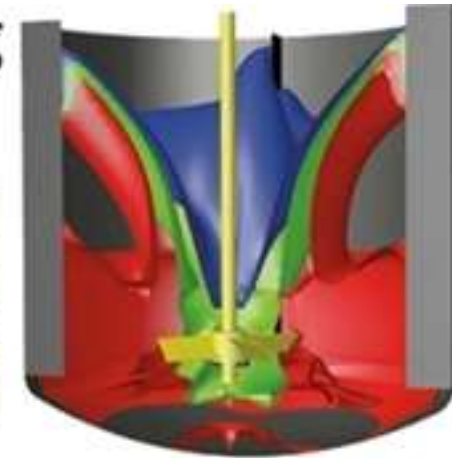
Chemical sprays



Heat exchangers



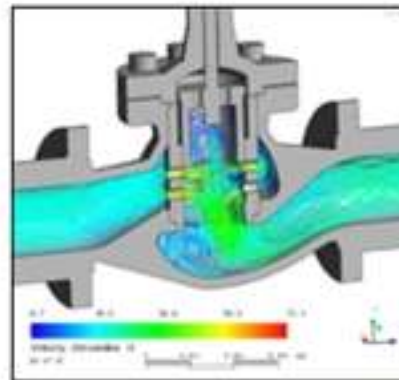
Dryers



Mixing tanks



Metal processing

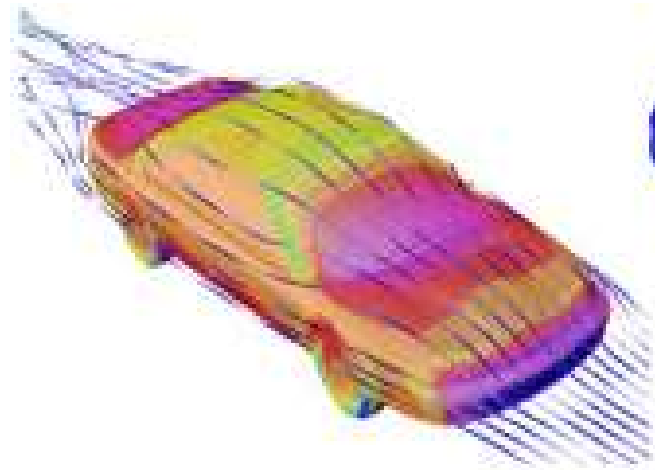


Valves, flow control

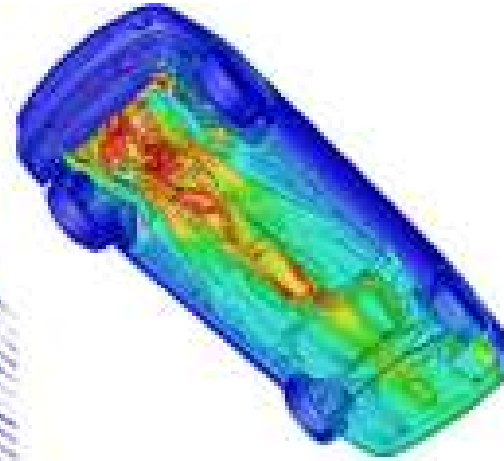


Separation and filtration

Automotive



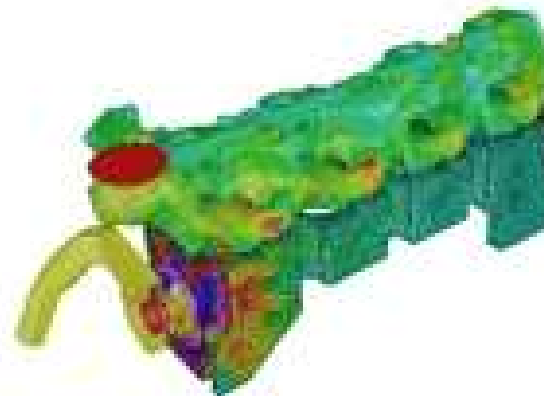
External Aerodynamics



*Undercarriage
Aerodynamics*

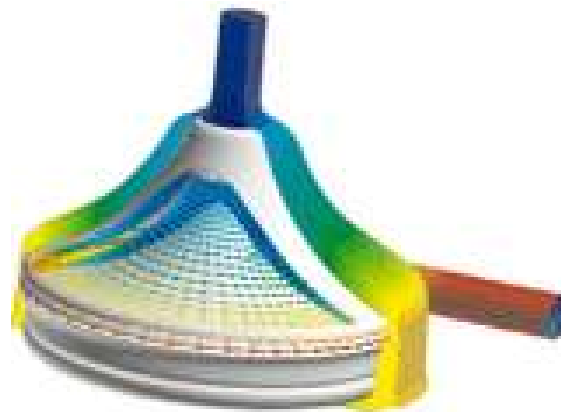


Interior Ventilation

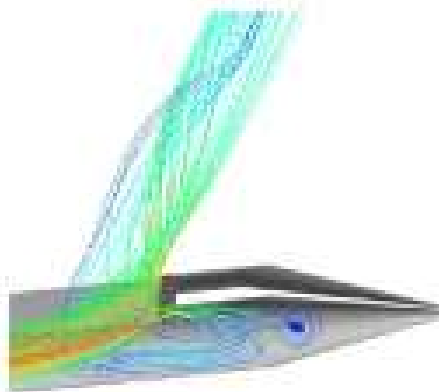


Engine Cooling

Medical



Medtronic Blood Pump

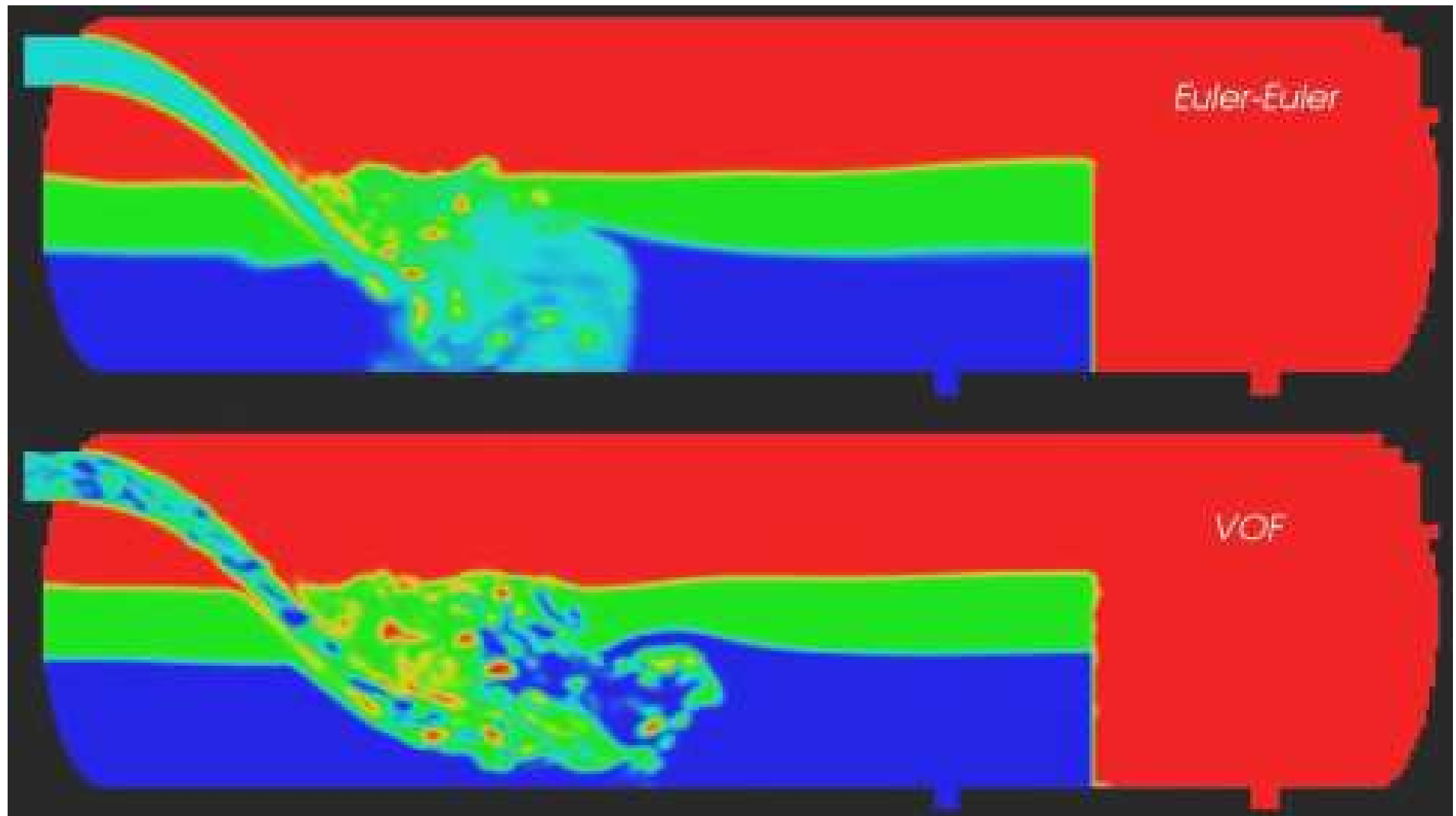


Spinal Catheter



Temperature and natural convection currents in the eye following laser heating.

Multiphase flows

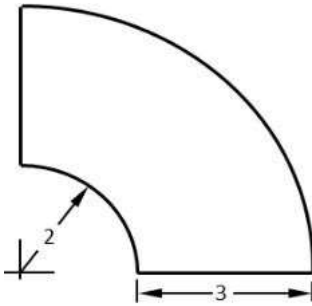


Oil- water separator

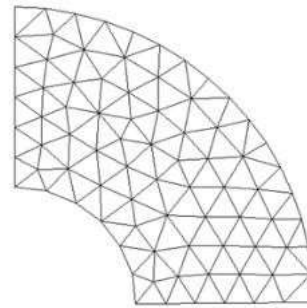


CFD SIMULATION PROCESS

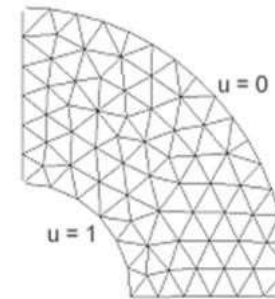
CFD process- Illustration



1. Build geometry



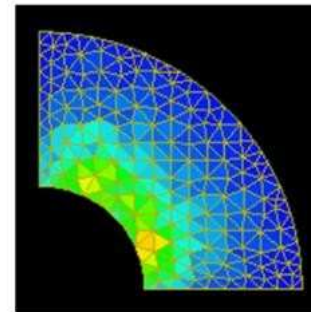
2. Mesh



3. Define boundary conditions

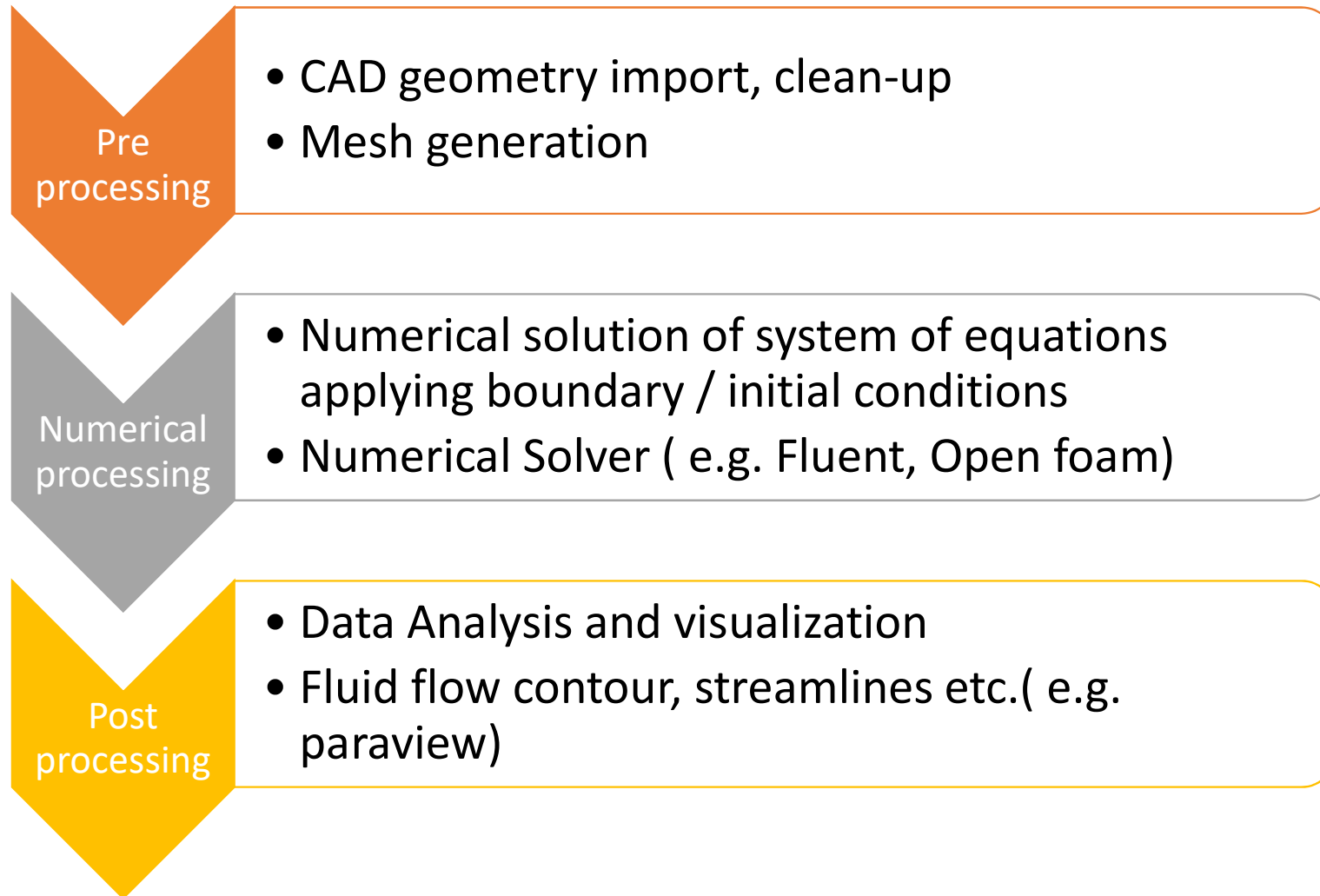


4. Compute



5. Visualize

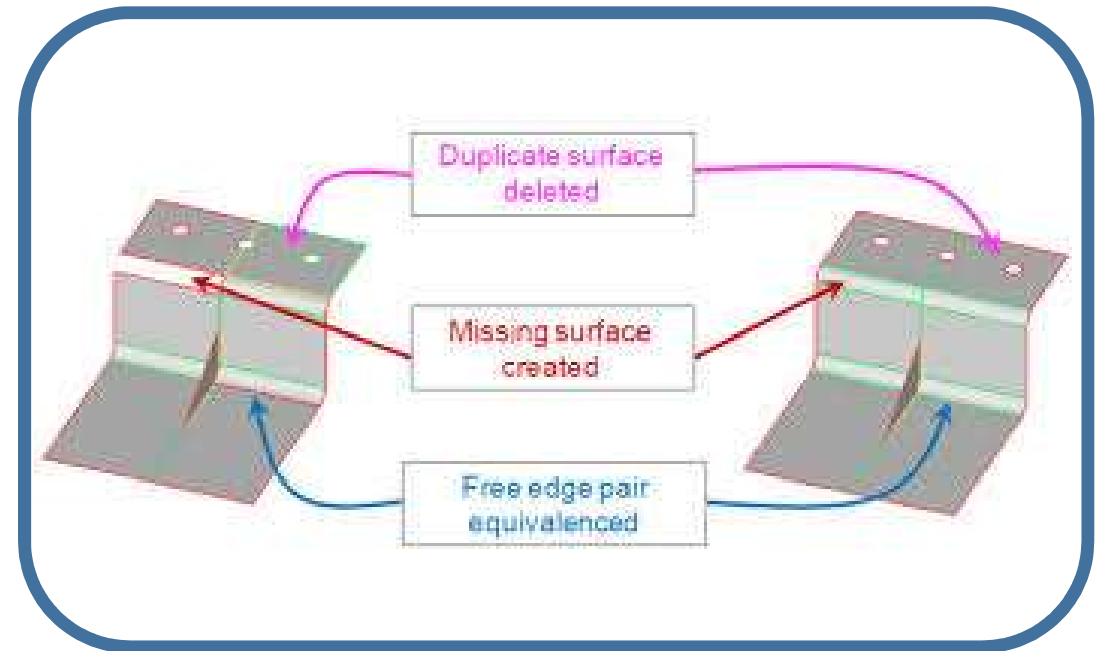
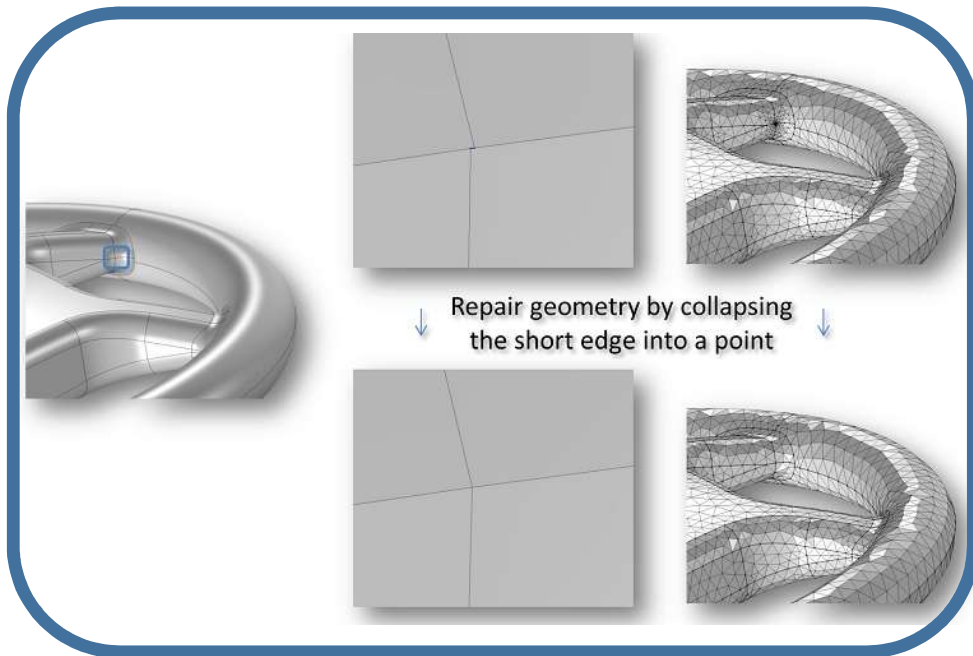
CFD process- Flow chart



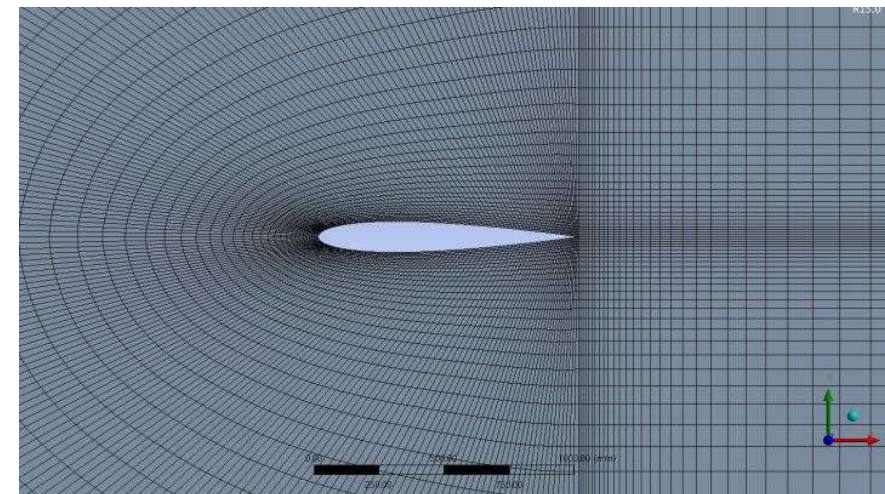
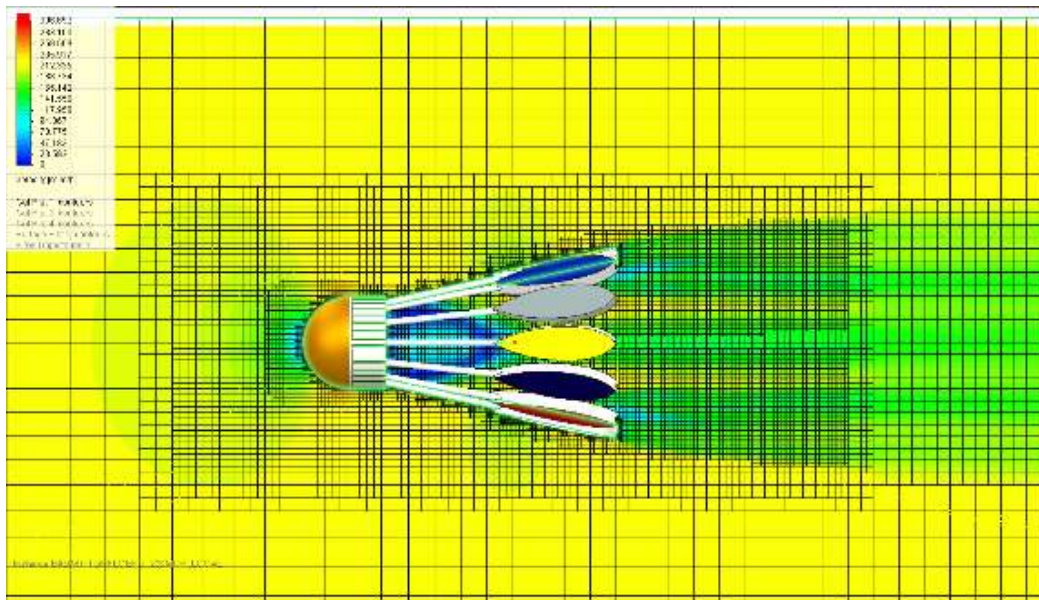
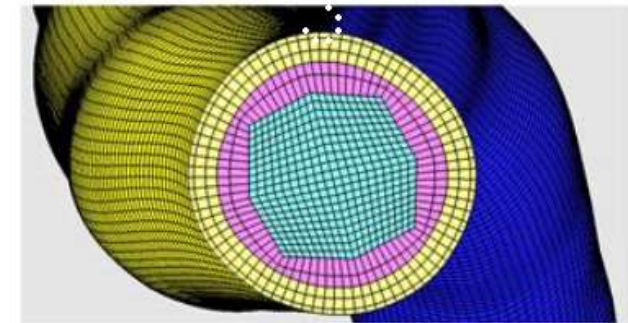
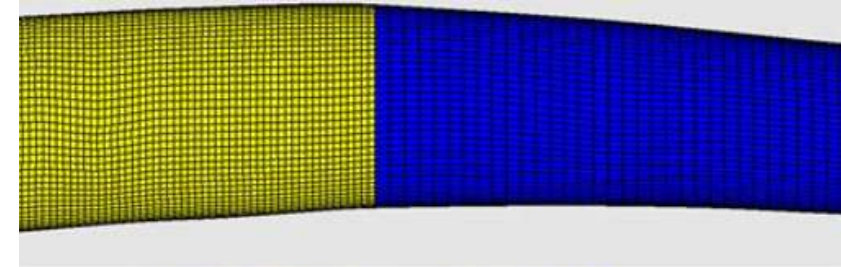
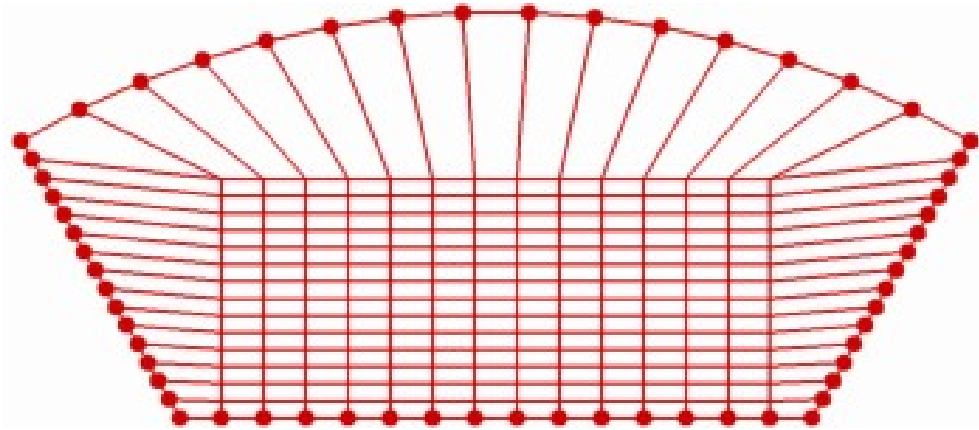


PRE PROCESSING STAGE

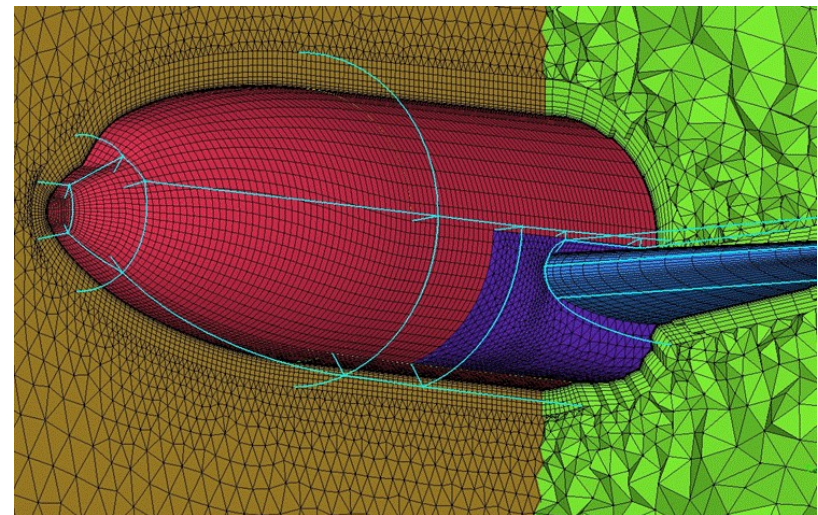
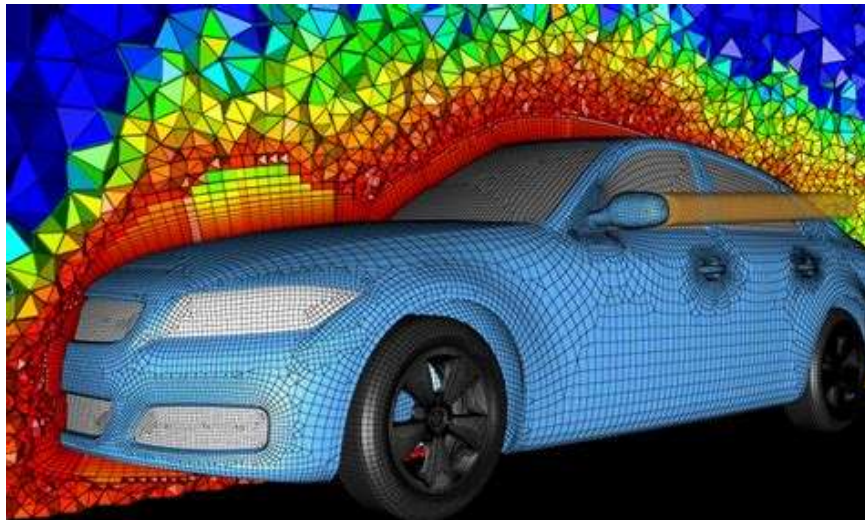
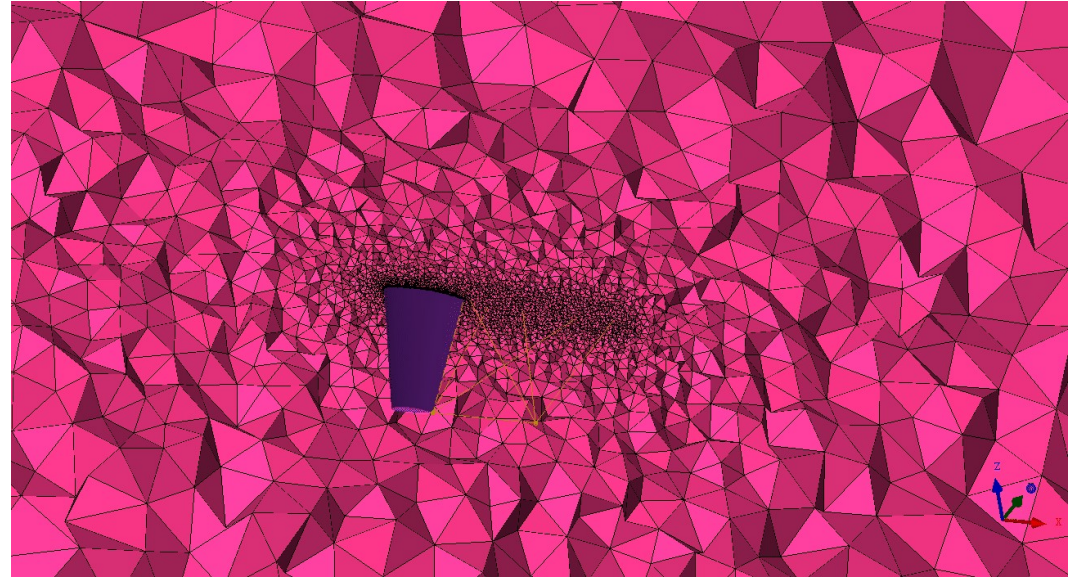
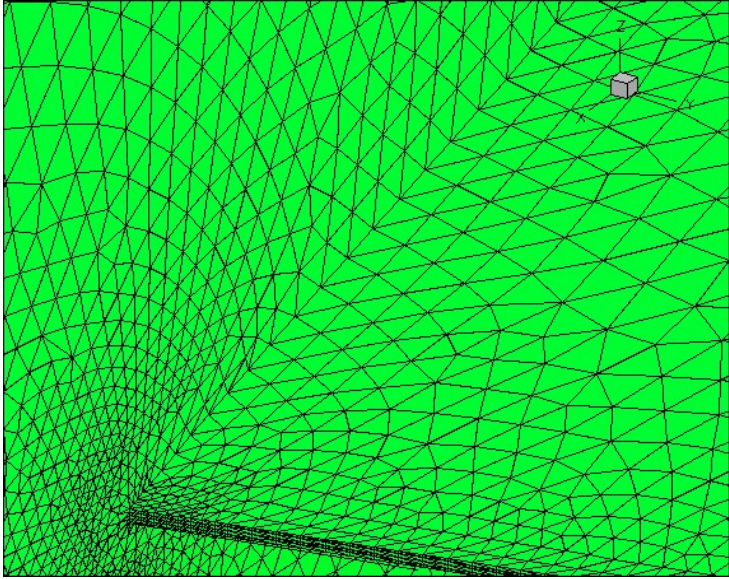
Geometry import & clean up – An illustration



Mesh generation – Structured mesh



Mesh generation – Unstructured mesh





NUMERICAL PROCESSING STAGE

Unknowns in the Governing Equations

- In the CFD simulation, it is required to solve numerically a set of Non-linear partial differential equations called the Navier- Stokes Equations.

- For example the governing equations for incompressible flow is given as,

Continuity eq.:
$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

x-mom.:
$$\rho\left(\frac{\partial u}{\partial t} + u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} + w\frac{\partial u}{\partial z}\right) = -\frac{\partial p}{\partial x} + \mu\nabla^2 u + \rho g_x$$

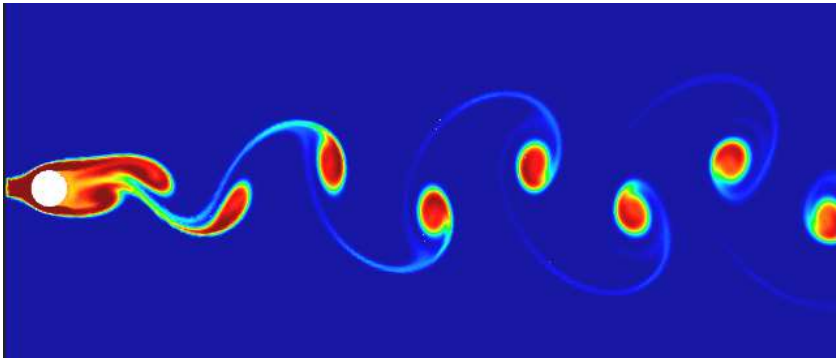
y-mom.:
$$\rho\left(\frac{\partial v}{\partial t} + u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y} + w\frac{\partial v}{\partial z}\right) = -\frac{\partial p}{\partial y} + \mu\nabla^2 v + \rho g_y$$

z-mom.:
$$\rho\left(\frac{\partial w}{\partial t} + u\frac{\partial w}{\partial x} + v\frac{\partial w}{\partial y} + w\frac{\partial w}{\partial z}\right) = -\frac{\partial p}{\partial z} + \mu\nabla^2 w + \rho g_z$$

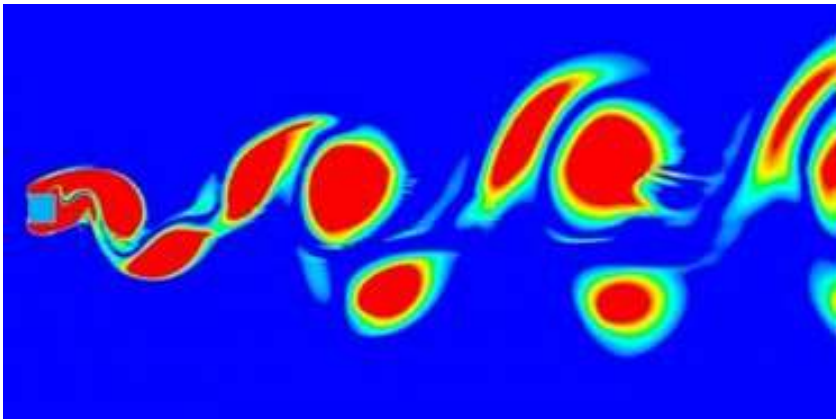
- These equations governs the laws of conservation of mass, momentum.
- The unknown includes the velocity and pressure of the fluid at several discrete points.
- There are several pressure and velocity correction based algorithms available to solve these equations (e.g. SIMPLE)

Types of fluid flow problems

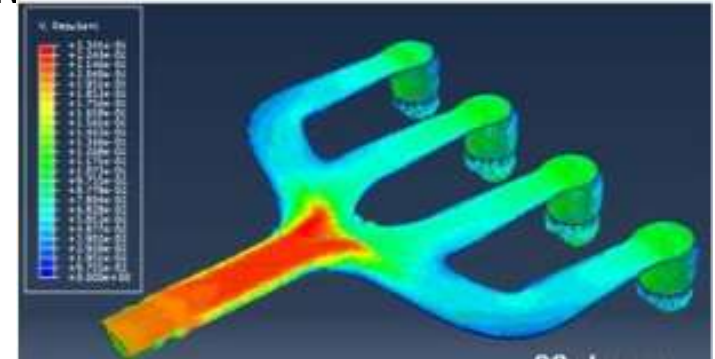
- In the CFD simulation, the fluid flow problems are broadly classified into external and internal flow problems.
- Further classification include steady or unsteady, compressible or incompressible, Laminar or Turbulent flow, one or two or three dimensional flows, natural or forced convection flows



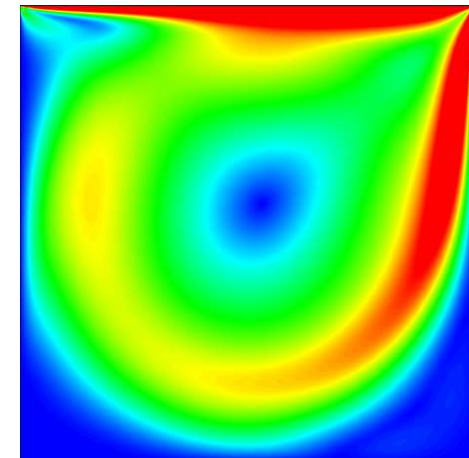
Unsteady external flow past a circular cylinder



Unsteady external flow past a square cylinder



Internal flow in a pipe

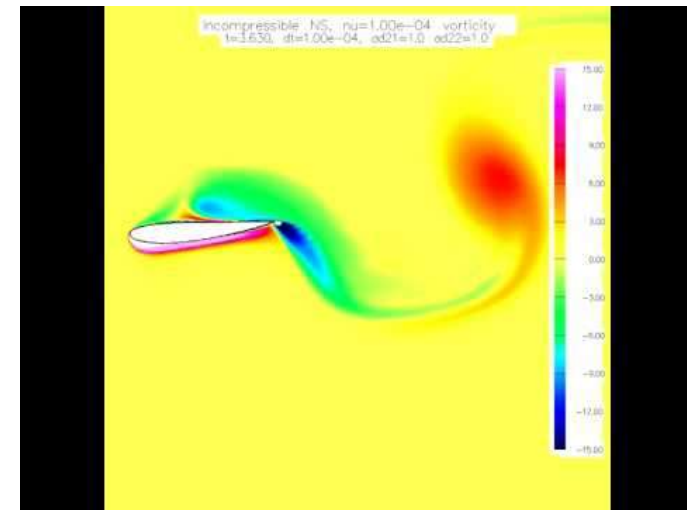
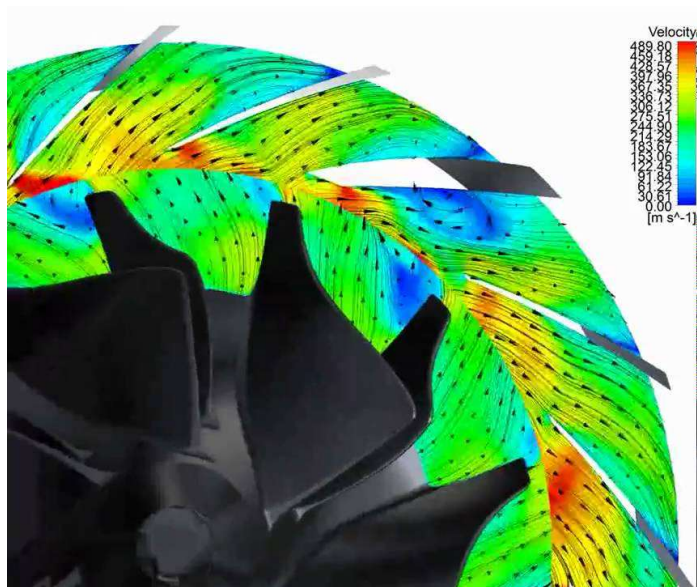
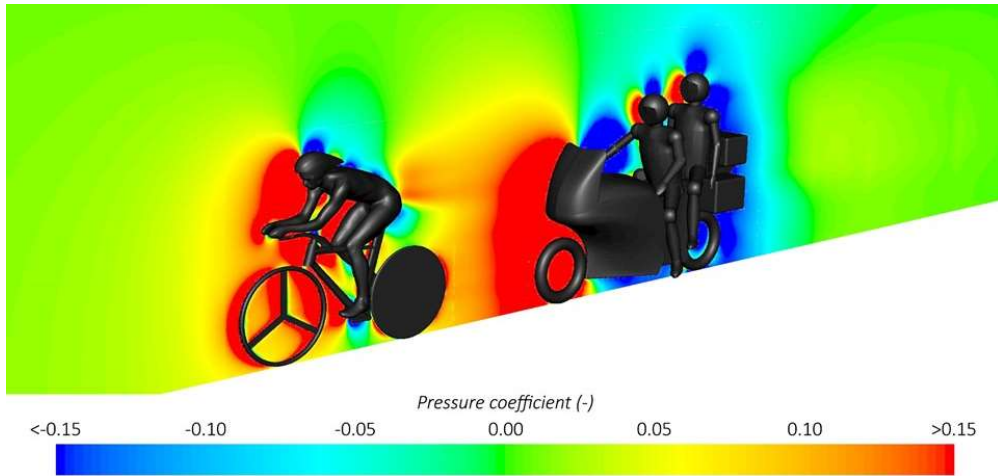


Flow inside a lid driven cavity



POST PROCESSING STAGE

Data Analysis & Visualization- An Illustration



The background of the slide is a light blue color with a dense pattern of small, realistic water droplets. The droplets vary in size and are scattered across the entire surface, giving it a textured, wet appearance.

CLASSIFICATION OF PARTIAL DIFFERENTIAL EQUATION CHARACTERISTICS

Characteristics of PDE systems

Consider the linear PDE system

$$A \frac{\partial^2 \phi}{\partial x^2} + B \frac{\partial^2 \phi}{\partial x \partial y} + C \frac{\partial^2 \phi}{\partial y^2} = 0$$

This system is said to be elliptic for the case $B^2 - 4AC < 0$.

It is parabolic if $B^2 - 4AC = 0$.

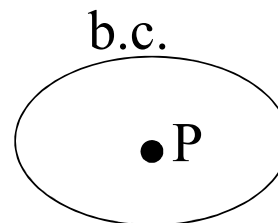
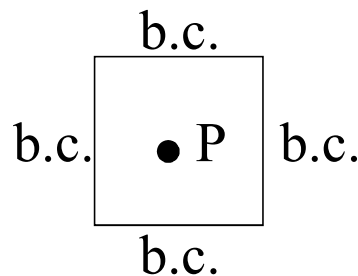
It is hyperbolic when $B^2 - 4AC > 0$.

Elliptic PDE

- Consider steady two dimensional heat conduction governed by the equation

$$k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) + Q = 0$$

- Here, $A = C = k$ and $B = 0$. Hence $B^2 - 4AC = -4k^2 < 0$.
- Therefore, the system is elliptic.
- For an elliptic PDE, the boundary conditions need to be given on a closed boundary.
- In other words, the boundary conditions all around influence the solution at a point



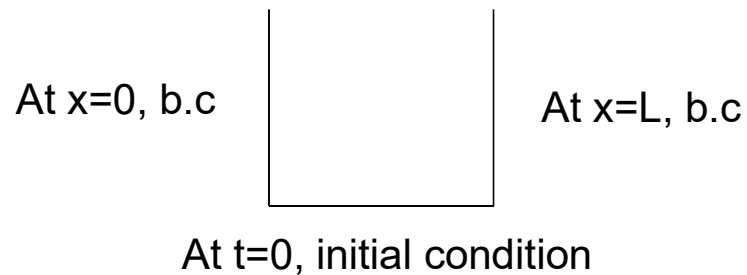
Boundary conditions for elliptic systems

Parabolic PDE

- Transient heat conduction problem which follows the governing equation

$$\frac{\partial T}{\partial t} = \frac{k}{\rho c} \frac{\partial^2 T}{\partial x^2} = \alpha \frac{\partial^2 T}{\partial x^2}$$

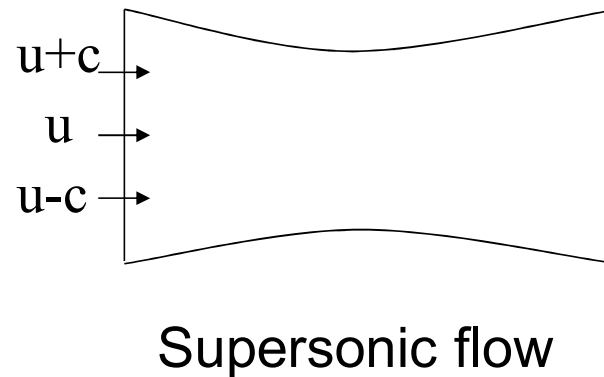
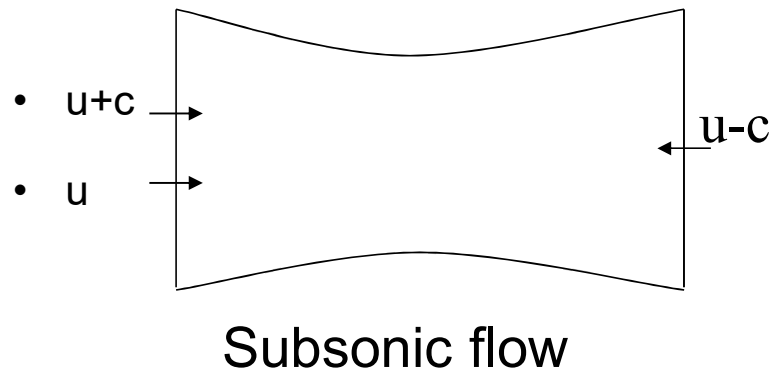
- Here, $A = \alpha$, $B = 0$ and $C = 0$.
- Hence, $B^2 - 4AC = 0$
- It is a parabolic system.
- For a parabolic system the conditions need to be specified as shown below.



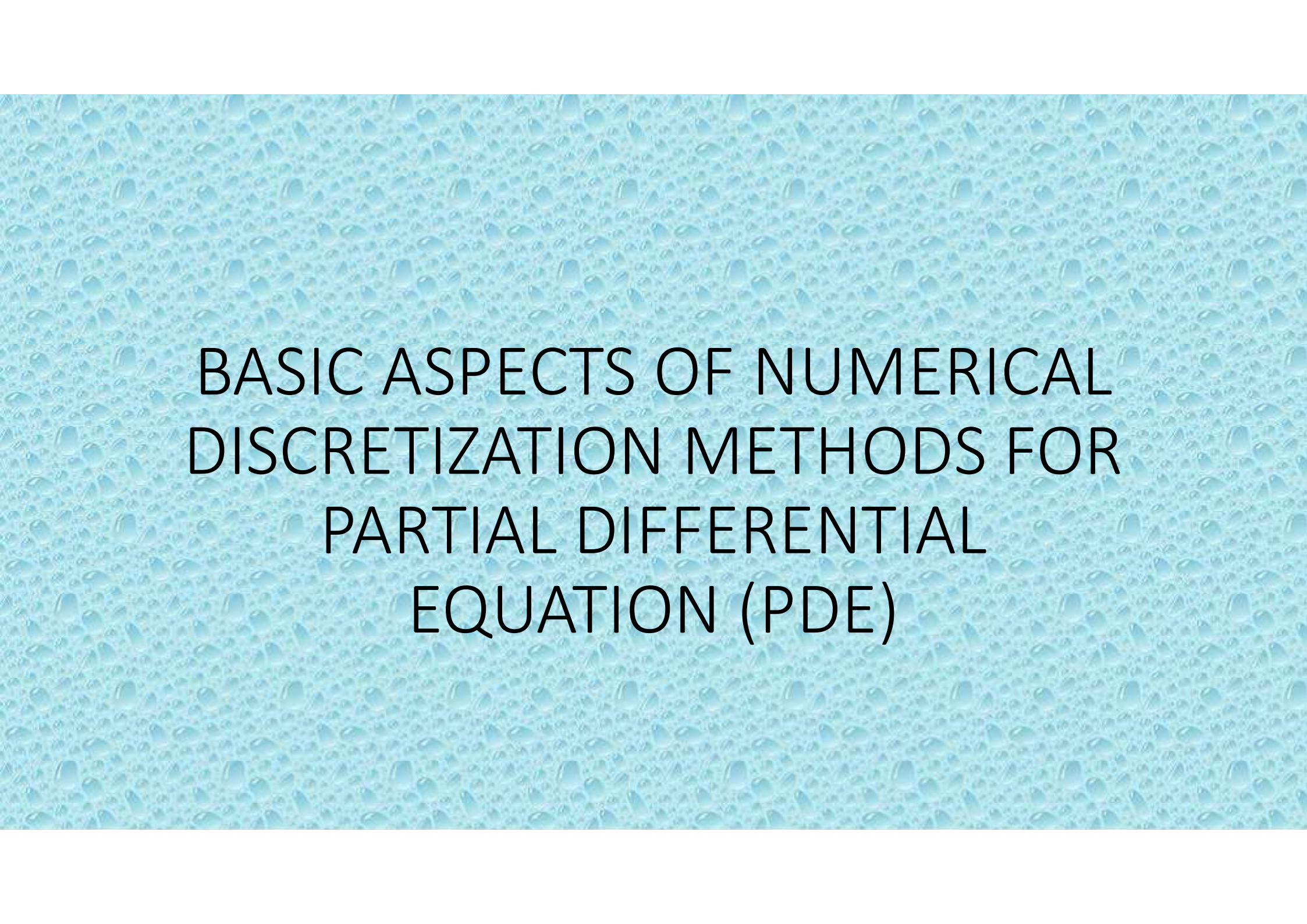
Hyperbolic PDE

- The wave equation $\frac{\partial^2 y}{\partial t^2} = c^2 \frac{\partial^2 y}{\partial x^2}$ is a hyperbolic system, with c denoting the acoustic speed.
- Here, $B = 0$ and $A = 1$, $C = -c^2$.
- Hence, $B^2 - 4AC = 0 - 4 \times 1 \times (-c^2) = 4c^2 > 0$.
- For a hyperbolic system, there are characteristic variables which determine the number of boundary conditions to be given.
- In the above case, the two characteristics $(x + ct)$ and $(x - ct)$ represent the solutions corresponding to the backward-and forward- propagating waves.

Boundary conditions for hyperbolic PDE



- A compressible flow has three characteristic velocities i.e. $u+c$, u , $u-c$.
- Depending on the number of characteristics crossing into the domain at the boundary, the b.c. are decided.



BASIC ASPECTS OF NUMERICAL DISCRETIZATION METHODS FOR PARTIAL DIFFERENTIAL EQUATION (PDE)

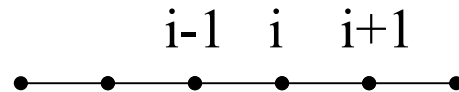
Types of Numerical Discretization Techniques

- FINITE DIFFERENCE METHOD
- FINITE VOLUME METHOD
- FINITE ELEMENT METHOD
- BOUNDARY ELEMENT METHOD
- SPECTRAL METHOD

Finite Difference Method

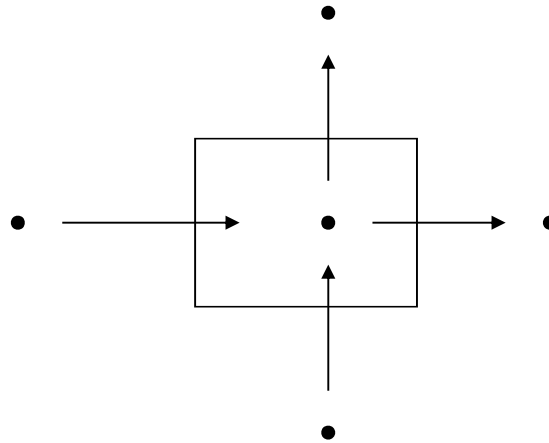
- In this method, differential equations are converted into difference expressions

$$\frac{dT}{dx} = \frac{T_i - T_{i-1}}{\Delta x} \quad \text{or} \quad \frac{T_{i+1} - T_i}{\Delta x}$$



Finite Volume Method

- Flux balance is applied for each cell.
- Heat flux in – Heat flux out = rate of thermal storage
- Fluxes are approximated using neighboring nodes



Finite Element Method

- While FDM & FVM are applied for flow/thermal problems, FEM was initially developed for structural problems.
- In this method, a large structure is divided into small elements and characteristic of each element is written as a matrix contribution.
- By adding contributions of all elements, we get matrix equation for the whole geometry.



APPLICATIONS OF FINITE DIFFERENCE METHOD

Taylor Series Expansions

$$T_{i-1} = T_i - \left(\frac{dT}{dx} \right)_i \Delta x + \left(\frac{d^2 T}{dx^2} \right)_i \frac{\Delta x^2}{2!} - \left(\frac{d^3 T}{dx^3} \right)_i \frac{\Delta x^3}{3!} + \dots + \left(\frac{d^n T}{dx^n} \right)_i \frac{(-\Delta x)^n}{n!} + O(\Delta x^{n+1})$$

$$T_{i+1} = T_i + \left(\frac{dT}{dx} \right)_i \Delta x + \left(\frac{d^2 T}{dx^2} \right)_i \frac{\Delta x^2}{2!} + \left(\frac{d^3 T}{dx^3} \right)_i \frac{\Delta x^3}{3!} + \dots + \left(\frac{d^n T}{dx^n} \right)_i \frac{\Delta x^n}{n!} + O(\Delta x^{n+1})$$

$$T_{i+2} = T_i + \left(\frac{dT}{dx} \right)_i (2\Delta x) + \left(\frac{d^2 T}{dx^2} \right)_i \frac{(2\Delta x)^2}{2!} + \left(\frac{d^3 T}{dx^3} \right)_i \frac{(2\Delta x)^3}{3!} \\ \dots + \left(\frac{d^n T}{dx^n} \right)_i \frac{(2\Delta x)^n}{n!} + O(\Delta x^{n+1})$$

$$T_{i-2} = T_i - \left(\frac{dT}{dx} \right)_i (2\Delta x) + \left(\frac{d^2 T}{dx^2} \right)_i \frac{(2\Delta x)^2}{2!} - \left(\frac{d^3 T}{dx^3} \right)_i \frac{(2\Delta x)^3}{3!} \\ + \left(\frac{d^n T}{dx^n} \right)_i \frac{(-2\Delta x)^n}{n!} + O(\Delta x^{n+1})$$

Derivative Approximations

$$\begin{aligned}\left(\frac{dT}{dx}\right)_i &= \frac{T_{i+1} - T_i}{\Delta x} - \left(\frac{d^2 T}{dx^2}\right)_i \frac{\Delta x}{2!} - \left(\frac{d^3 T}{dx^3}\right)_i \frac{\Delta x^2}{3!} \\ &= \frac{T_{i+1} - T_i}{\Delta x} + O(\Delta x)\end{aligned}$$

$$\begin{aligned}\left(\frac{dT}{dx}\right)_i &= \frac{T_i - T_{i-1}}{\Delta x} - \left(\frac{d^2 T}{dx^2}\right)_i \frac{\Delta x}{2!} + \left(\frac{d^3 T}{dx^3}\right)_i \frac{\Delta x^2}{3!} \\ &= \frac{T_i - T_{i-1}}{\Delta x} + O(\Delta x)\end{aligned}$$

$$\left(\frac{dT}{dx}\right)_i = \frac{T_{i+1} - T_{i-1}}{2 \Delta x} + O(\Delta x^2)$$

Derivative Approximation

$$4T_{i+1} - T_{i+2} = 3T_i + 2\left(\frac{dT}{dx}\right)_i \Delta x + 0(\Delta x^3)$$

$$\left(\frac{dT}{dx}\right)_i = \frac{4T_{i+1} - T_{i+2} - 3T_i}{2\Delta x} + 0(\Delta x^2)$$

$$T_{i+1} + T_{i-1} = 2T_i + 2\left(\frac{d^2T}{dx^2}\right)_i \frac{\Delta x^2}{2!} + 2\left(\frac{d^4T}{dx^4}\right)_i \frac{\Delta x^4}{4!} + \dots$$

$$\left(\frac{d^2T}{dx^2}\right)_i = \frac{T_{i+1} + T_{i-1} - 2T_i}{\Delta x^2} + 0(\Delta x^2)$$

Estimation of Error

$$\varepsilon_i^k = T(x_i, t^k) - T^*(x_i, t^k)$$

$$\varepsilon_i^k \propto \Delta x_i^2 \quad \text{and} \quad \varepsilon_i^k \propto \Delta t^k$$

$$\varepsilon = O(\Delta x^2, \Delta t)$$



NUMERICAL ALGORITHM TO SOLVE NAVIER STOKES EQUATION- PRESSURE CORRECTION APPROACH

VELOCITY-PRESSURE FORMULATION

CONTINUITY EQUATION

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

X-MOMENTUM EQ. (FOR UPDATING U VELOCITY):

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = - \frac{1}{\rho} \frac{\partial p}{\partial x} + \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\}$$

Y-MOMENTUM EQ. (FOR UPDATING V VELOCITY):

$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = - \frac{1}{\rho} \frac{\partial p}{\partial y} + \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\}$$

SIMPLE METHOD

Semi- IMplicit Pressure Linked Equation Solver-- SIMPLE

X-mom.:
$$\frac{\partial u}{\partial t} = - \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \Big| ^n - \frac{1}{\rho} \frac{\partial p}{\partial x} \Big| ^{n+1} + \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\} \Big| ^n$$

$$u^{n+1} = u^n - \Delta t. \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \Big| ^n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial x} \Big| ^{n+1} + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\} \Big| ^n$$

Y-mom.:
$$\frac{\partial v}{\partial t} = - \left(u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \Big| ^n - \frac{1}{\rho} \frac{\partial p}{\partial y} \Big| ^{n+1} + \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\} \Big| ^n$$

$$v^{n+1} = v^n - \Delta t. \left(u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \Big| ^n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial y} \Big| ^{n+1} + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\} \Big| ^n$$

VELOCITY CORRECTION EQUATION- x momentum

$$u^{n+1} = u^n - \Delta t. \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \Big|_n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial x} \Big|^{n+1} + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\} \Big|_n$$

$$u^* = u^n - \Delta t. \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \Big|_n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial x} \Big|_n^* + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right\} \Big|_n$$

$$u^{n+1} - u^* = -\Delta t. \left(\frac{1}{\rho} \frac{\partial p}{\partial x} \right)^{n+1} + \Delta t. \left(\frac{1}{\rho} \frac{\partial p}{\partial x} \right)^*$$

VELOCITY CORRECTION EQUATION- y momentum

$$v^{n+1} = v^n - \Delta t. \left(u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \Bigg|_n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial y} \Bigg|^{n+1} + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\} \Bigg|_n$$

$$v^* = v^n - \Delta t. \left(u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \Bigg|_n - \Delta t. \frac{1}{\rho} \frac{\partial p}{\partial y} \Bigg|_n^* + \Delta t. \frac{\mu}{\rho} \left\{ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right\} \Bigg|_n$$

$$v^{n+1} - v^* = -\Delta t. \left(\frac{1}{\rho} \frac{\partial p}{\partial y} \right)^{n+1} + \Delta t. \left(\frac{1}{\rho} \frac{\partial p}{\partial y} \right)^*$$

PRESSURE CORRECTIONS

Define

$$u' = u^{n+1} - u^* \quad v' = v^{n+1} - v^* \quad p' = p^{n+1} - p^*$$

It can be shown that

$$u' = -\frac{\Delta t}{\rho} \frac{\partial p'}{\partial x} \quad v' = -\frac{\Delta t}{\rho} \frac{\partial p'}{\partial y}$$

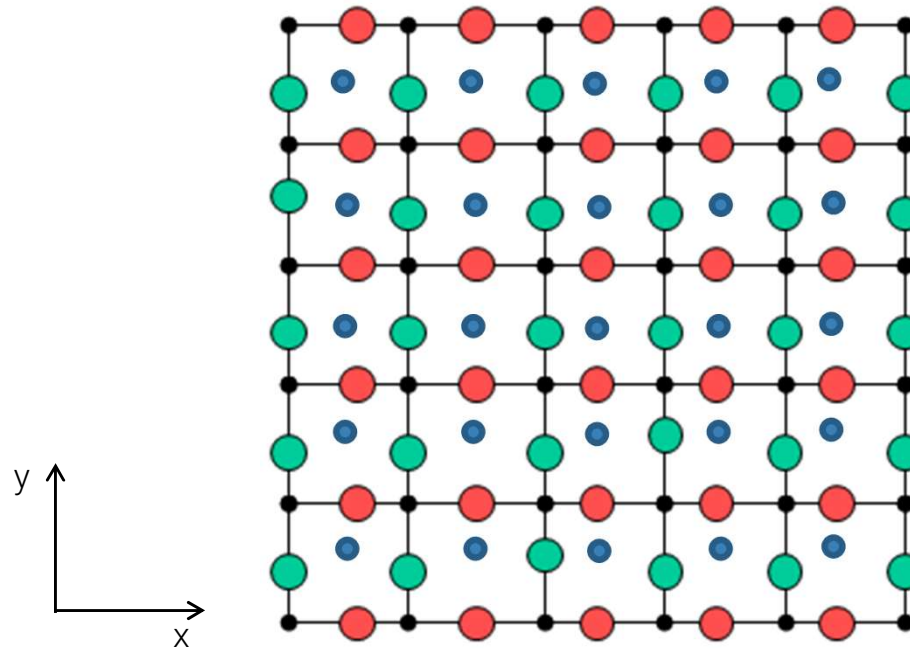
Substituting for velocity & pressure corrections, we get

$$\frac{\partial^2 p'}{\partial x^2} + \frac{\partial^2 p'}{\partial y^2} = -\frac{\rho}{\Delta t} \left(\frac{\partial u'}{\partial x} + \frac{\partial v'}{\partial y} \right) = \frac{\rho}{\Delta t} \left(\frac{\partial u^*}{\partial x} + \frac{\partial v^*}{\partial y} \right)$$

Step Involved In SIMPLE

- At the start of a time step, assume a guess pressure field p^*
- Solve momentum equations to get guess velocities u^* and v^* at each node
- Using u^* and v^* calculate continuity residue at each point
- From continuity equation residue, solve for pressure correction p' at each node
- Using p' solve for velocity corrections
- Update variables as $p^{n+1}=p^*+p'$, $u^{n+1}=u^*+u'$, $v^{n+1}=v^*+v'$
- And go to next time step

Staggered & Collocated Mesh



	Staggered	Semi-Staggered	Collocated
●	V- velocity	V- velocity	-
●	U- velocity	U- velocity	-
●	Cell vertices	Pressure (Cell vertices)	-
●	Pressure (Cell centers)	Cell centers	U,V- velocities, Pressure

Staggered Mesh Procedure

- Pressure nodes are taken as the main nodes.
- x-velocity (u) nodes are shifted by $dx/2$ with reference to pressure nodes .
- and y-velocity (v) nodes are shifted by $dy/2$ with reference to pressure nodes.
- Such a staggered mesh avoids odd-even decoupling (chequer-board configuration) between velocities & pressures .